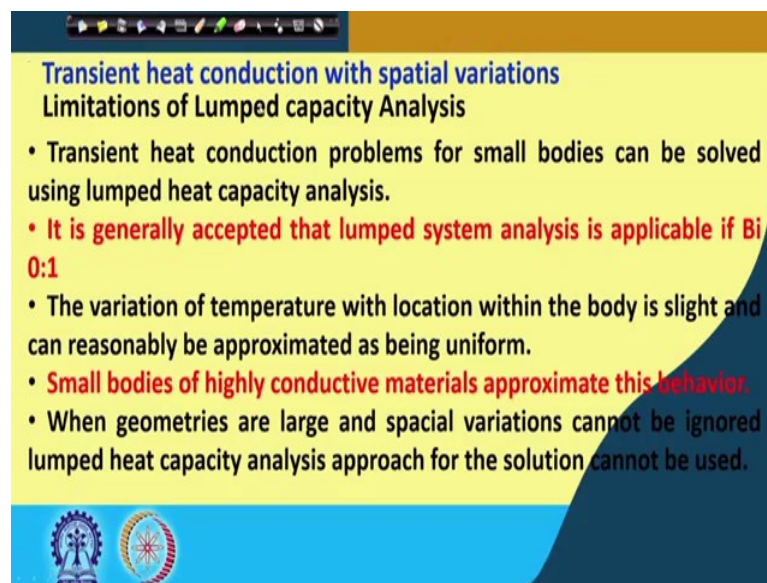


**Thermal Operations in Food Process Engineering: Theory and Applications**  
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**Lecture - 25**  
**Heisler Chart**

Good Morning. We said in the previous class, this is of course lecture number 25, 'right'; lecture number 25 and we will do here that Heisler Chart, 'right'. So, this we said in the previous class if you remember, that we have done some non-dimensional parameters like theta, like capital X, like alpha, like, like Fourier number, like Biot number. These things we had said in the previous class. So, today we shall or in this class we shall use those for the transient heat transfer, 'right'.

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**Transient heat conduction with spatial variations**  
**Limitations of Lumped capacity Analysis**

- Transient heat conduction problems for small bodies can be solved using lumped heat capacity analysis.
- **It is generally accepted that lumped system analysis is applicable if  $Bi < 0.1$**
- The variation of temperature with location within the body is slight and can reasonably be approximated as being uniform.
- **Small bodies of highly conductive materials approximate this behavior.**
- When geometries are large and spacial variations cannot be ignored lumped heat capacity analysis approach for the solution cannot be used.

So, here we see that the transient conduction heat transfer with spatial variations, are that is limitations of the lumped capacity analysis, 'right'. So, if this was the solid and if all along this solid, the temperature is uniform then this can be under lump system; but if in this temperature here is  $T_1$ , here is  $T_2$ , here is  $T_3$ , here is  $T_4$ , like that, then; that means, it is spatial variations, 'right', so that is not handled by the lumped system analysis.

So, in that transit heat conduction problems for small bodies can be solved using lumped system analysis; where it is generally accepted that the lumped system analysis is done if

Biot number is less than 1, 'right'. So, that less than symbol has not come here or it has gone out of here, so it should be that less than 0.1.

So, the variation of temperature with location within the body and this can be, and can and reasonably be approximated as being uniform and small bodies of highly conducting material approximate this behaviour, when geometries are large and spatial variations can be, can spatial variations cannot be ignored lumped heat capacity analysis approach solution cannot be employed, 'right'. So, that cannot be employed when your system is big or where Biot number is not .1, 'right'.

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The slide contains the following content:

- Graph:** A graph showing temperature profiles in a slab of thickness  $L$  for different times  $t=0$ ,  $t=t_1$ , and  $t \rightarrow \infty$ . The initial temperature is  $T_i$  and the ambient temperature is  $T_\infty$ . The heat transfer coefficient is  $h$ .
- Governing Equation:** Analytical Solution of one dimensional generalized condition involves this governing equation along with necessary boundary conditions such as:
 
$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$$
- Boundary Conditions:**

$$\frac{\partial T(0,t)}{\partial x} = 0$$

$$\text{and } -k \frac{\partial T(L,t)}{\partial x} = h\{T(L,t) - T_\infty\}$$
- Note:** The analytical solution of these situations involve infinite series and implicit equations, which are difficult to evaluate. One-term approximation solutions of above equation are presented in graphical form, known as the transient temperature charts, or Heisler Charts

So, here we see that we have taken a lumped system, where you can have a uniformity in the like the temperature is like this, 'right'. So, analytical solution of one dimensional generalized condition involves the governing equations along with the necessary

boundary conditions, 'right', and these are that is the governing equation

$$\frac{\partial^2 T}{\partial x^2} = \frac{1}{\alpha} \frac{\partial T}{\partial t}$$

$$\text{and } -k \frac{\partial T(L,t)}{\partial x} = h\{T(L,t) - T_\infty\}$$

boundary condition is , 'right'.

$$\frac{\partial T(0,t)}{\partial x} = 0$$

So, lumped system analysis cannot do this for that we need the analytical solution of these situations, involving infinite series and implicit equations which are very difficult to evaluate. One term approximation or solutions of above equation are presented graphically and these graphics are known as the transient temperature charts or simply Heisler charts, 'right'.

So, this we have given an example for that. So, where lumped system cannot be applied there, analytical solution becomes very clumsy or very tricky or very difficult to handle. So, in those cases graphical solution was proposed by great scientist called Heisler, 'right' and according to him the charts are named as Heisler chart, 'right'.

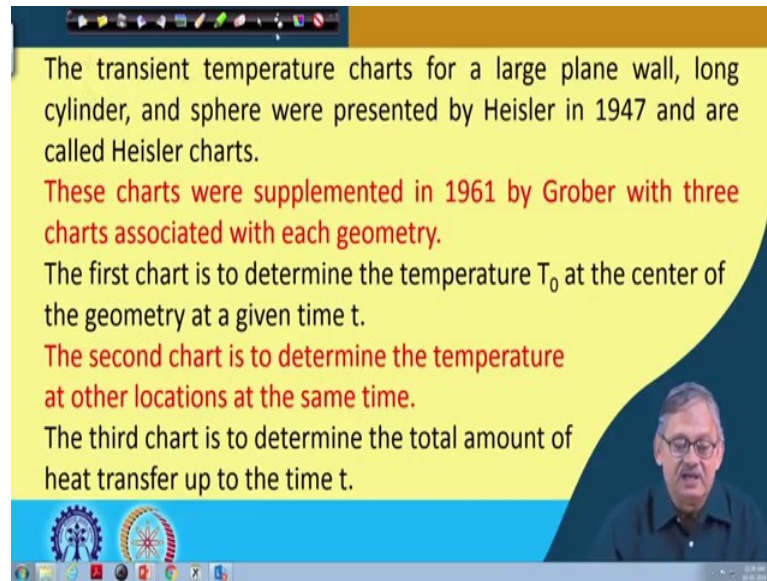
So, there in earlier if you remember we had given geometry, say that this is one dimensional. So, this is the other two dimensions are become much larger compared to this that is why we can say it to be one dimensional. So, its axis is at the centre, 'right'.

So, heat is being transferred in the x direction this is 0 to L 'right'. So, if it is 0 to L it is 0 to minus L which we had shown earlier, 'right'; that means, that whatever is happening in this half it is the mirror image of this half 'right' this is identical, 'right'. So, that is what is reflected through this at different time  $t_3$   $t_2$   $t_1$  the width and  $t_0$  this is at  $T_0$  it was the initial condition

So, after this is at  $t$  infinity environmental condition with heat transfer coefficient of  $h$ , 'right' the same is true at this side also because it being cylindrical. So, initial condition was  $T$  is  $T_i$ , 'right' and we have solution where this can be that after  $t$  infinity this temperature inside can also be  $T_\infty$

So, from  $T_i$  to how it went to  $t$  infinity was like this, 'right'. So, this is and you see that this up is nothing, but the mirror image of this up, 'right'. So, that is what explicitly will utilize and this was done by the Heisler and that is why the charts are known as Heisler chart, 'right'.

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The transient temperature charts for a large plane wall, long cylinder, and sphere were presented by Heisler in 1947 and are called Heisler charts.

These charts were supplemented in 1961 by Grober with three charts associated with each geometry.

The first chart is to determine the temperature  $T_0$  at the center of the geometry at a given time  $t$ .

The second chart is to determine the temperature at other locations at the same time.

The third chart is to determine the total amount of heat transfer up to the time  $t$ .

The transient temperature charts for large plane wall, long cylinder and sphere were presented by Heisler in 1947 and are called Heisler charts, 'right'. Then these charts were supplemented in 1961 by Grober with three charts associated with each of them each geometry rather. The first chart is to determine the temperature  $T_0$  at the centre of the geometry at a given time  $t$ . Then the second chart is to determine the temperature at other locations at the same time, 'right' and the third chart is to determine the total amount of heat transfer up to the time  $t$ , 'right'.

So, let me go back a little that the first chart is to determine the temperature  $T_0$  at the centre of the geometry at a given time  $t$ . The second chart is to determine the temperature at other locations at the same time and the third chart is to determine the total amount of heat transferred up to the time  $t$ , 'right'.

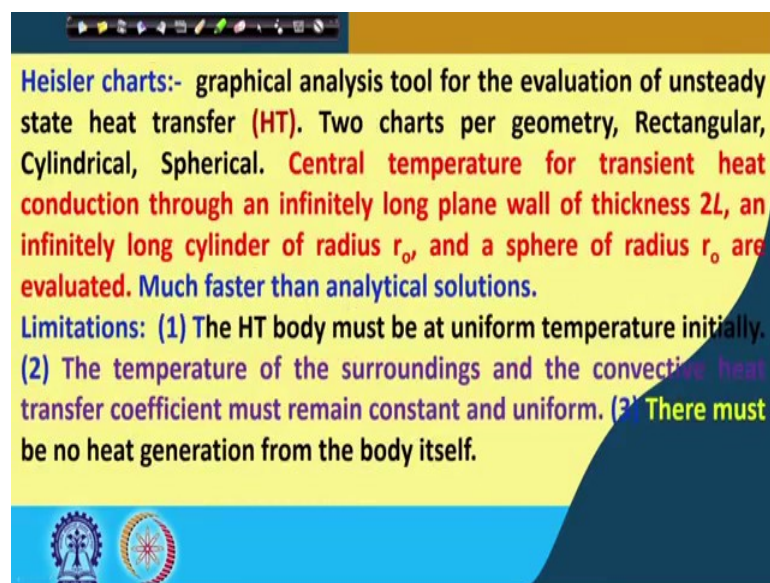
This just now we had shown that picture that this we can say that here, 'right'. So, the things which we have just said that at any time  $t$  what is the center temperature say this one, 'right' at any time  $t$  what is the center temperature say this one, 'right' this is at  $x$  is equal to 0, 'right'.

Then we also said that at the same time that is at time  $t$  is equal to  $t$  what is the temperature at other location, 'right' that also can be found out and what is the heat being transferred  $Q$ , 'right'. These three though you are getting this solution or not solution this pictorial view showing that how the temperatures are being distributed if it be a

cylindrical solid or if it be a cylindrical substance, then how the temperature is distributed that you have shown.

So, Heisler charts does that. So, at  $x=0$  what is the temperature at time  $t$  at  $x$  is equal to any  $x$  what is the temperature at time  $t$  and total  $Q$  that can be found out from the Heisler chart, 'right' and this along with the other along with the other researcher Heisler made it and subsequently he was supplemented by Grover, 'right' and we have said what are the three charts, ok.

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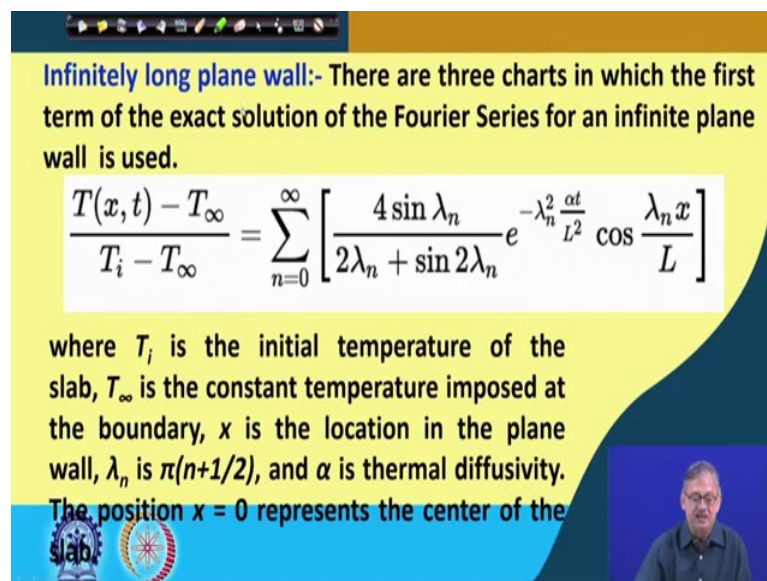


Now, Heisler charts which are graphical analysis tool for the evolution of unsteady state heat transfer or HT. Two charts per geometry, rectangular, cylindrical or spherical are available. Central temperature for transient heat conduction through an infinite long plane wall of thickness  $2L$  and infinitely long cylinder of radius  $r_0$  and a sphere of radius  $r_0$  are evaluated by that those charts. Much faster than analytical solutions. Of course, the solutions are much quicker than analytical.

Limitations are what number one the heat transfer body must be at uniform temperature initially that is the first limitation. Say then, the temperature of the surroundings our surroundings and the convective heat transfer coefficient must remain constant and uniform and third one that there must be there must be no heat generation from the body itself.

So, these are the three, these are the three defects or limitations of the Heisler chart. Number one that the initial temperature must be uniform all along this is really an approximation because in much geometry or many materials this may not be true. Then the second is the temperature of the surroundings and the convective heat transfer coefficient must be constant and uniform and third is that there must not be any generation of internal heat which we have seen earlier, 'right'. So, these three are the limitations of the Heisler chart, 'right'.

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**Infinitely long plane wall:-** There are three charts in which the first term of the exact solution of the Fourier Series for an infinite plane wall is used.

$$\frac{T(x, t) - T_{\infty}}{T_i - T_{\infty}} = \sum_{n=0}^{\infty} \left[ \frac{4 \sin \lambda_n}{2\lambda_n + \sin 2\lambda_n} e^{-\lambda_n^2 \frac{\alpha t}{L^2}} \cos \frac{\lambda_n x}{L} \right]$$

where  $T_i$  is the initial temperature of the slab,  $T_{\infty}$  is the constant temperature imposed at the boundary,  $x$  is the location in the plane wall,  $\lambda_n$  is  $\pi(n+1/2)$ , and  $\alpha$  is thermal diffusivity. The position  $x = 0$  represents the center of the slab.

So, now we come for the infinitely long plane, 'right'. There are three charts in which the first term of the exact solution of the Fourier series for an infinite plane wall is used, 'right'. So, that solution says like this that

$$\frac{T(x, t) - T_{\infty}}{T_i - T_{\infty}} = \sum_{n=0}^{\infty} \left[ \frac{4 \sin \lambda_n}{2\lambda_n + \sin 2\lambda_n} e^{-\lambda_n^2 \frac{\alpha t}{L^2}} \cos \frac{\lambda_n x}{L} \right]$$

So, this is an analytical solution, 'right'. Of course, the terms are like this  $T_i$  is the initial temperature of this body  $T_{\infty}$  is the constant temperature imposed at the boundary,  $x$  is the location in the plane any location in the plane wall and  $\lambda_n$  is  $\pi$  times  $n$  plus 1 by 2 and  $\alpha$  is the thermal diffusivity.

The position  $x$  is equal to 0 represents the center, 'right' of the slab, 'right'; the center of the slab is the position  $x$  is equal to 0, 'right'. So, if this is a solution by analytical

method you see solving such an equation is how much time consuming and how much energy as well what devotions you have to do. Whereas, the same thing or a similar one you know, because here you have already said that this is true for infinitely long plane wall; so that means, this is the plane and the other two other three sides like this and that these sides are infinite, 'right'.

So, but we are only concerned with this side, 'right'. If that be true we can find out what is the temperature distribution or from the non-dimensional theta x which is

$$\theta_0 = \frac{T(0,t) - t_\infty}{T_i - T_\infty}$$

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The first chart for the plane wall is plotted using 3 different variables. Plotted along the vertical axis of the chart is dimensionless temperature at the midplane.  $\theta_0 = \frac{T(0,t) - t_\infty}{T_i - T_\infty}$ . Plotted along the horizontal axis is the Fourier Number,  $Fo = \frac{\alpha t}{l^2}$ . The curves within the graph are a selection of values for the inverse of the Biot Number, where,  $Bi = \frac{hl}{k}$ ,  $k$  is the thermal conductivity of the material, and  $h$  is the heat transfer coefficient.

Handwritten notes:  $Bi = \frac{hl}{k}$ ,  $\frac{1}{Bi}$ , and a sketch of a wall with arrows indicating heat transfer.

Then we can say that the first chart for the plane wall is plotted using the three different variables. Plotted along the vertical axis of the chart; along the vertical axis of the chart and the chart is dimensionless temperature at the mid plane that is theta 0 is T<sub>0</sub> t minus T<sub>i</sub> minus T infinity, 'right'. Plotted along the horizontal x axis horizontal axis is the Fourier number 'right' which we call this y and x axis.

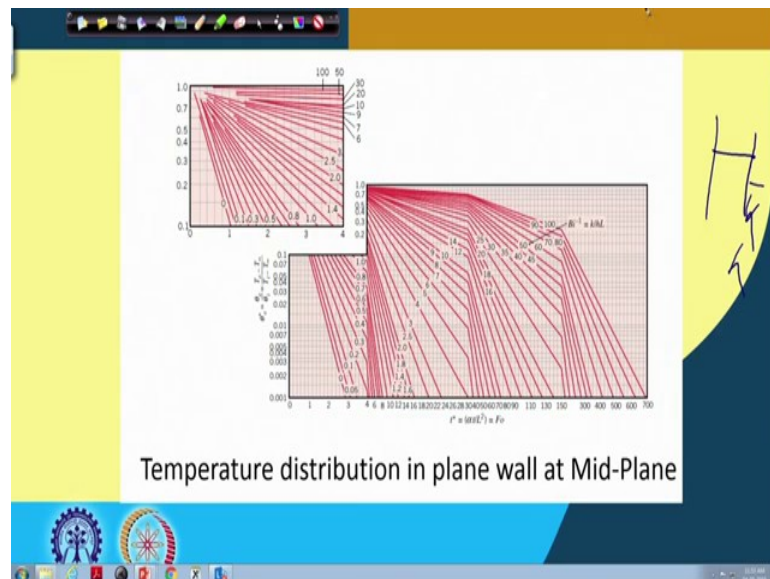
This number is Fourier number, 'right' and this is theta, 'right' and where this theta was

equal to  $\theta_0 = \frac{T(0,t) - t_\infty}{T_i - T_\infty}$  and Fourier number is  $Fo = \frac{\alpha t}{l^2}$ , 'right'.

So, this is known. So, the curves within the graph are a selection of values for the inverse of the Biot number, 'right'. So,  $1/Bi$ ;  $Bi$  was,  $h/k$ , 'right'. So, inverse of  $Bi$  is the selection of the curves say this is one curve, this is another curve, this is another curve. So, this is for  $1/Bi$ , this is for another one by  $Bi$  like that, 'right'.

So, selection of the curves within the graph are selection of the values for the inverse of Biot number where  $Bi$  is  $h/k$  and  $k$  is the thermal conductivity of the material and  $h$  is the heat transfer coefficient. So, we have explained how the Heisler charts are being prepared or were prepared, 'right'.

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Now, if we look at one of them that temperature distribution plane wall at mid plane, 'right'. So, this is saying temperature distribution in plane wall at mid-plane that is we have a plane wall like this, at the mid plane like this, what is the temperature distribution? So, that with time is shown, 'right'; where it is Fourier number is  $\alpha t$  by

$L$  square, 'right'. Fourier number is

$$Fo = \frac{\alpha t}{l^2} \quad \text{and} \quad \theta_0 = \frac{T(0,t) - t_\infty}{T_i - T_\infty}, \text{ 'right' and this is a}$$

log plot, 'right' 0 to 1 for theta, 'right' and Fourier number ranging 0, 100, 700 like that.

So, these are  $1/Bi$  that is  $k/hL$ ; these are  $1/Bi$  that is  $k/hL$ , 'right'. So, if  $1/Bi$  is known and if Fourier number is known, then you can select that graph and find out from here



what the value of theta is and in that your initial temperature is known, your environmental temperature is known and you can find out what is the value of  $T_0$ , 'right'.

This is a graphical. Only other than those assumptions which we have made, rather other than those limitations which we have shown or talked about there is also one more thing which has to be said explicitly, that the prime defect is that those are inherent defective cannot do anything, 'right'. That the body having uniform initial temperature that you cannot make any deals, any change, 'right'.

So, we in most of the cases this is not true, 'right'. The reason being that there will be always a temperature gradient in the body that is true, but we are assuming for this that the body has uniform initial temperature fine. Then we also said that the environmental temperature  $T_\infty$  and  $h$  they are constant and uniform fine and the third thing we said that that assumption, the third thing which is said was that, if I remember correctly that the third thing which we said was this, 'right'; there must not be any sorry I forgot that there must not be any internal heat generation, 'right'.

Internal energy generation there must not be; you remember in one dimensional heat transfer in conduction rather; we had shown the lot many cases where internal generation of heat was there. So, this is not valid where you can apply this kind of Heisler chart, 'right'.

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The second chart is used to determine the variation of temperature within the plane wall for different Biot Numbers. The vertical axis is the ratio of a given temperature to that at the centerline

$$\frac{\theta}{\theta_0} = \frac{T(x,t) - T_\infty}{T(0,t) - T_\infty}$$

where the  $x/L$  curve is the position at which  $T$  is taken.

The horizontal axis is the value of  $Bi^{-1}$ .

So, what other these are inherent; other thing which is very much that is you have found out Bi, sorry you have found out Bi fine. Your  $1/Bi$  is how much you know? So, whatever value say 3, 'right'. So, you can find out corresponding to that 3 this curve, 'right'.

Now, you also found out  $Fo = \frac{\alpha t}{l^2}$ . Say that value you have found out to be, say, 18,

'right'. So, from there if you can come to this point and see this point then you get the value of theta 0.

Here is the possibility of error, that what you can see that this line and this line they are coming the value of theta which you have taken say 0.085 or something that you may say 0.085, your friend may say 0.086 or your may say 0.084, 'right'. So, that error is unavoidable. This is a human error or personal error, 'right', but not the system error this is human error or personal error, 'right'.

So, there can be this and this charge normally, which are available in the book difference Heisler charts in big, big posters perhaps they are not. The reason being nowadays many, many software have come up which may help you to solve such kind of solution, such kind of situations, much quicker and in a much errorless way.

So, that is why people have not taken or given that much consideration to enlarge those Heisler charts. Yes, if Heisler chart should have been enlarged like that you can also with minimum error, you can also find out. The same is true I hope you have done not in this class in some other classes that humidity chart or simply we called psychrometric chart, 'right' that psychometry also is another human error involved situation, 'right'.

So, what we can find out that  $\theta_0$  at;  $\theta_0$  at any time t, 'right' that  $\theta_0$  at any time t corresponding to Fourier number Fo, 'right' and also  $1/Bi$ . So, you have to know the Biot number, 'right'. So, if that is known then other charts which are available are like this. The second chart is used to determine the variation of temperature within the plane wall of different Biot numbers, 'right'. The vertical axis is the ratio of the given temperature to that at the center temperature.

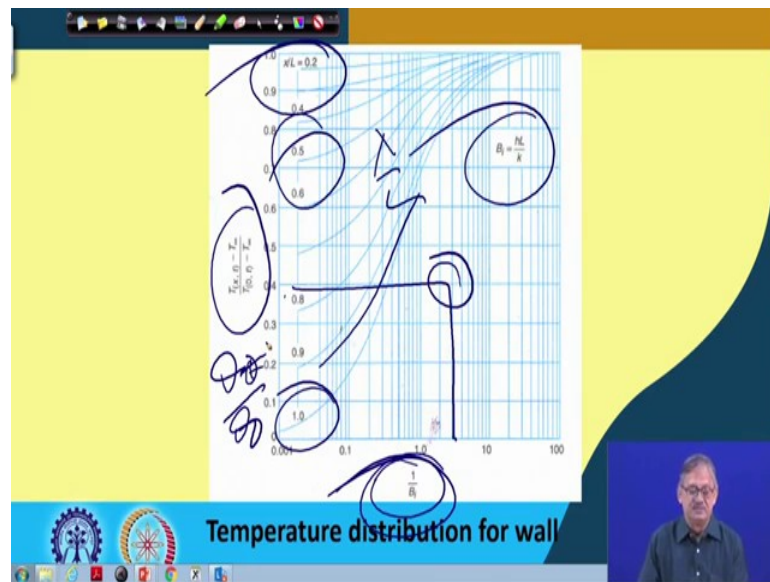
So, that is  $\theta_0 = \frac{T(0,t) - T_\infty}{T_i - T_\infty}$  and  $x/L$  is the curve is the position at which  $T$  is taken,

$$\theta_0 = \frac{T(0,t) - T_\infty}{T_i - T_\infty}$$

‘right’ and horizontal axis is the value of  $1/Bi$ , ‘right’. So,  $Bi$  inverse is the horizontal value the y axis is  $\theta_0$ , ‘right’. This is also non-dimensional and that is  $\{T(x,t) - T_\infty / T(0,t) - T_\infty\}$  and you have already found out the Biot number.

So, inverse of Biot numbers is there and you are finding out  $x/L$  for different  $x/L$ ; obviously, this will  $x$  will be maximum at 1 and  $x$  will be minimum at  $x/L$  is 0, ‘right’.

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So, this is the second curve. Let us look into that. The temperature distribution for the wall and there it appears, you see that  $1/Bi$  is there and  $\theta_t / \theta_0$  that it  $\{T(x,t) - T_\infty / T(0,t) - T_\infty\}$  is there and  $Bi$  is  $1/hL/k$  and  $x/L$  is that 0.02 0.03 0.456 like that up to 1, ‘right’.

So, from this curve knowing  $1/Bi$  you know where you are, then you know you are you also know  $x/L$ . So, from there you know this point and you can get that  $\theta_t / \theta_0$ , ‘right’  $\theta_t / \theta_0$  that you can find out easily, ‘right’.

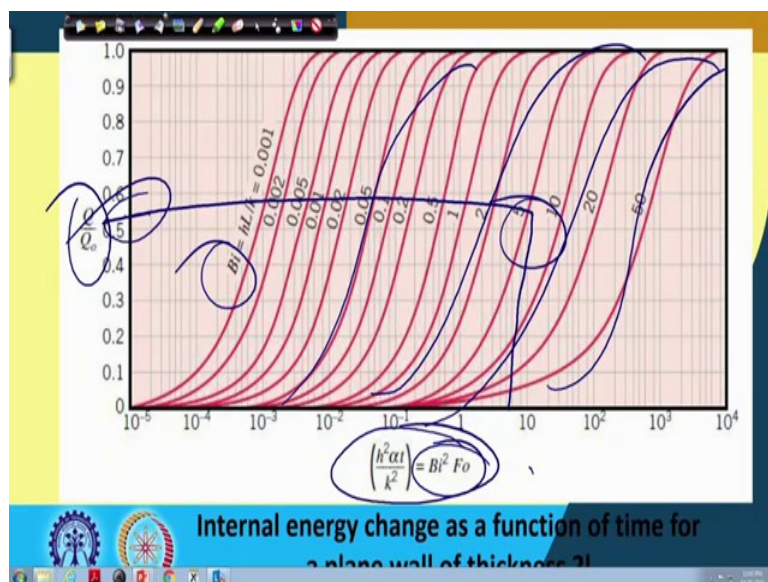
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The third chart in each set was supplemented by Gröber in 1961 and this particular one shows the dimensionless heat transferred from the wall as a function of a dimensionless time variable. The vertical axis is a plot of  $Q/Q_0$ , the ratio of actual heat transfer to the amount of total possible heat transfer before  $T = T_\infty$ . On the horizontal axis is the plot of  $(Bi^2)/Fo$ , a dimensionless time variable.



So, let us look into the third very quickly because time is almost over. So, the third one is the third chart in each said was supplemented by Grober in 1961 where you are plotting  $Q$  by  $Q_0$ ; the ratio of actual heat transfer to the amount of total possible heat transfer; for  $T$  is  $T_\infty$  on the horizontal axis is the plot  $Bi^2/Fo$  a dimensionless time is also being plotted or is plotted.

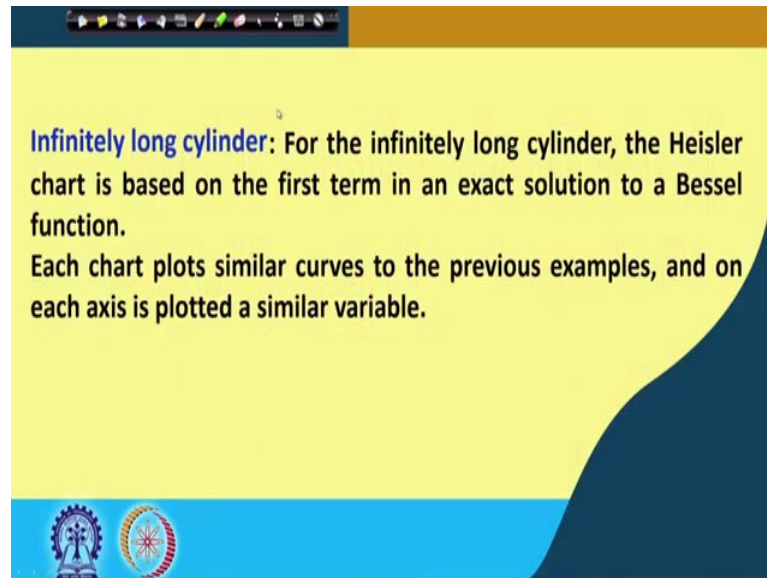
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Then, that if we look at it appears like this. So, here it is this is like that  $(h_s^2 \alpha \times t) / k^2$  that is  $Bi^2/F_0$  on the x axis this is  $Q/Q_0$  on the y axis; and this is  $Bi = hL/k$ .

So, these lines will tell us for different Bi what is the value of corresponding Bi into  $F_0$  square versus this Bi we can get the value of  $Q/Q_0$ , 'right'. So, hopefully this is what we can look at that, 'right'.

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So, afterwards perhaps this class is over subsequently we will do again maybe some numerals, some numerical solutions, numerical means which number we will try to find out, ok.

Thank you.