

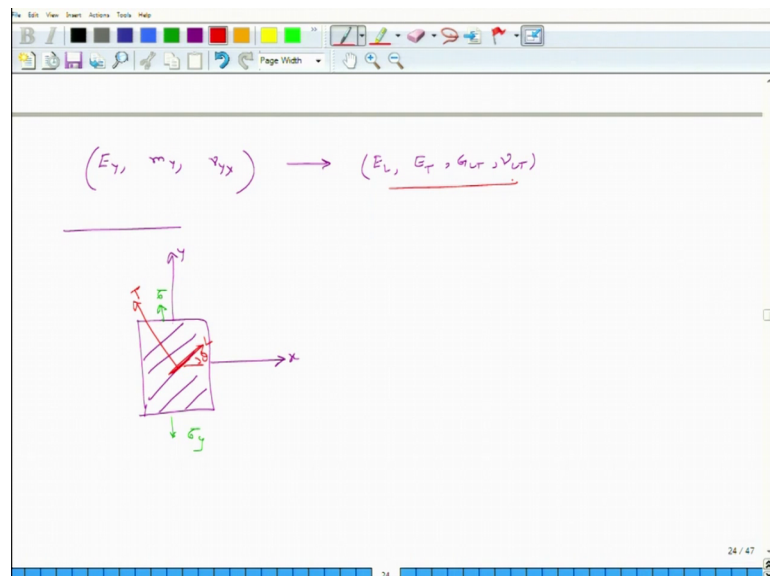
**Introduction to Composites**  
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**Lecture – 48**  
**Transformation of Engineering Constants – Part II**

Hello, welcome to Introduction to Composites. Today is the last day of the ongoing week and over last five days what we have done successfully is we have developed mathematical relationships between stress and strain for especially orthotropic lamina. We have also exhibited and shown the mathematical relationships between stress and strain for a generally orthotropic lamina and in the last class what we did accomplished was we started developing relationships between elastic constants as applicable to generally orthotropic lamina which are  $E_x$ ,  $E_y$ ,  $G_{xy}$ ,  $m_x$ ,  $m_y$ ,  $\mu_{xy}$ ,  $\mu_{yx}$  and the elastic constants for especially orthotropic lamina  $E_L$ ,  $E_T$ ,  $G_{LT}$  and  $\nu_{LT}$  and what we have shown is that specifically that  $E_x$ ,  $\nu_{xy}$  and  $m_x$  can be exhibit expressed in terms of  $E_L$ ,  $E_T$ ,  $G_{LT}$  and  $\nu_{LT}$ .

Today, we will develop the remaining relationships and at the end of the course at the end of this day, we will show that all the elastic constants for generally orthotropic lamina can be reduced in terms of simple functions which are applicable for especially orthotropic lamina.

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So, today, the next phase is that we will develop relationships between  $E_y$ ,  $m_y$  and  $\nu_{yx}$  or actually  $\nu_{xy}$ . We will develop function relationships between these things to special orthotropic material properties which are  $E_L$ ,  $E_T$ ,  $G_{LT}$  and  $\nu_{LT}$ .

So, we will develop these three relations.

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①  $E_y$  as a function of  $E_L, E_T, G_{LT}, \nu_{LT}$

In  $(x, y)$  :  $(\sigma_x, 0, 0)$   
 In  $(L, T)$  :  $(\sigma_L, \sigma_T, \tau_{LT})$

$$\begin{Bmatrix} \sigma_L \\ \sigma_T \\ \tau_{LT} \end{Bmatrix} = \sigma_x \begin{Bmatrix} \cos^2 \theta \\ \sin^2 \theta \\ -\sin \theta \cos \theta \end{Bmatrix}$$

$$\begin{aligned} \epsilon_L &= \frac{\sigma_L}{E_L} - \frac{\nu_{LT}}{E_T} \sigma_T \\ \epsilon_T &= -\nu_{LT} \frac{\sigma_L}{E_L} + \frac{\sigma_T}{E_T} \end{aligned}$$

$$\begin{aligned} \epsilon_L &= \frac{\sigma_x \cos^2 \theta}{E_L} - \frac{\nu_{LT}}{E_T} \sigma_x \sin^2 \theta \\ \epsilon_T &= -\nu_{LT} \frac{\sigma_x \cos^2 \theta}{E_L} + \frac{\sigma_x \sin^2 \theta}{E_T} \end{aligned}$$

For the first set of relationships we had loaded the material in the x direction. Now, what we do is we load the material in the y direction and we will get similar relations. So, here what we have is we have this material and this is my x direction, this is the L direction, this is my transverse direction, excuse me and this is my y direction and instead of loading this sample in the x direction, I applied a pure tensile load in the y direction.

So, I apply  $\sigma_y$  here the angle between L and x axis still is theta. So, this angle is still theta and if we exactly follow the process which we had discussed earlier, then we end up with the relationships between  $E_y$ ,  $m_y$  and  $\gamma_{yx}$  and these terms all the other steps are similar are very much the same. So, we do not have to repeat it. So, what I will do is, I will directly write down the relations.

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The slide shows three equations derived from stress-strain relations in a rotated coordinate system:

$$\frac{1}{E_y} = \frac{\sin^4 \theta}{E_L} + \frac{\cos^4 \theta}{E_T} + \frac{1}{4} \left\{ \frac{1}{G_{LT}} - \frac{2\nu_{LT}}{E_L} \right\} \sin^2 2\theta \quad D3$$

$$\frac{\nu_{yx}}{E_y} = \frac{\nu_{TL}}{E_T} - \frac{1}{4} \left[ \frac{1}{E_L} + \frac{1}{E_T} + \frac{2\nu_{LT}}{E_L} - \frac{1}{G_{LT}} \right] \sin^2 2\theta \quad D5$$

$$\gamma_{xy} = \sin 2\theta \left[ \nu_{LT} + \frac{E_L}{E_T} - \frac{E_L}{2G_{LT}} - \sin^2 \theta \left\{ 1 + 2\nu_{LT} + \frac{E_L}{E_T} - \frac{E_L}{G_{LT}} \right\} \right] \quad D6$$

So, the first relation is for  $E_y$ , Young's modulus in  $y$  direction, so, this is  $1$  over  $E_y$  and that equals  $\sin^4 \theta$  over  $E_L$  plus  $\cos^4 \theta$  over  $E_T$  plus  $1$  over  $4$  and I have  $1$  over  $G_{LT}$  minus  $2\nu_{LT}$  divided by  $E_L$   $\sin^2 2\theta$ . So, this is the relation for  $E_y$  and to get to this relation we have to follow exactly the same process which we did earlier.

The next relation is for  $\nu_{yx}$ . So,  $\nu_{yx}$  actually before  $\nu_{yx}$  we will express the relation for  $\gamma_{yx}$ . So,  $\gamma_{yx}$  is equal to  $\nu_{TL}$  over  $E_T$  and this is  $1$  over  $E_y$  on the right side minus  $1$  by  $4$ . So, it is  $\nu_{TL}$  over  $E_T$  minus  $1$  over  $4$  plus  $1$  over  $E_L$  plus  $1$  over  $E_T$  plus  $2\nu_{LT}$  divided by  $E_L$  minus  $1$  over  $G_{LT}$   $\sin^2 2\theta$ . So, this is the second relation.

And, the third relation, there is little confusion on the left side, so, I will just make it explicit. So, this is  $\nu_{yx}$  divided by  $E_y$  and then the cross coefficient  $\gamma_{xy}$  equals  $\sin 2\theta$  times this entire thing  $\nu_{LT}$  plus  $E_L$  over  $E_T$  minus  $E_L$  over  $2G_{LT}$  minus  $\sin^2 2\theta$ , oh actually I am sorry,  $\sin^2 \theta$  times  $1$  plus  $2\nu_{LT}$  plus  $E_L$  over  $E_T$  minus  $E_L$  over  $G_{LT}$ . So, these are the three relations and so, we will call them  $D3$ ,  $D4$ ,  $D5$  and  $D6$  and when you compare  $D3$ ,  $D4$ ,  $D5$ ,  $D6$  with their counterparts  $D1$ ,  $D2$ ,  $D3$ , what you find are some equivalences.

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Handwritten equations on a whiteboard:

$$\gamma_{xy} = -m_x \frac{\sigma_x}{E_L}$$

$$m_y = -\gamma_{xy} \frac{E_L}{\sigma_y}$$

And, that is gamma xy equals minus m x sigma x over E L and you also see that m y equals minus gamma xy times E L over sigma y, I mean these are the basic definitions. So, we use these definitions to develop the relations mentioned earlier.

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Handwritten diagram and equations on a whiteboard:

Diagram: A square element with axes  $x$ ,  $y$ ,  $L$ , and  $T$ . The  $x$  and  $y$  axes are horizontal and vertical respectively. The  $L$  and  $T$  axes are rotated by an angle  $\theta$  relative to the  $x$  and  $y$  axes. The square is shaded with diagonal lines.

Equations:

$$\sigma_x = 0$$

$$\sigma_y = 0$$

$$\tau_{xy} \neq 0$$

$$\sigma_L = 2 \tau_{xy} \sin \theta \cos \theta$$

$$\sigma_T = -2 \tau_{xy} \sin \theta \cos \theta$$

$$\tau_{LT} = \tau_{xy} (\cos^2 \theta - \sin^2 \theta)$$

Finally, we will develop an expression for  $G_{xy}$ , we will develop an expression for  $G_{xy}$ . So, what do we do, we again go back to our basic concept. We have an orthotropic composite lamina. The lamina is loaded in this direction is not especially orthotropic and so, this is my  $x$  axis, this is my  $y$  axis, this is my  $L$  axis and this is my  $T$  axis.

So, the angle between x and L axis is theta and here since we are interested in finding  $\epsilon_{xy}$ , we subject it to pure shear stress, positive shear stress. So, this is our positive shear stress is applied like this. So, this is  $\tau_{xy}$ . So, in this case  $\sigma_x$  is equal to 0,  $\sigma_y$  is equal to 0 and  $\tau_{xy}$  is not equal to 0. So, the first thing we do is we express  $\sigma_L$ ,  $\sigma_T$  and  $\tau_{LT}$ . So, we say from those stress transformation equations, tensor equations,  $\sigma_L$  is equal to  $2 \tau_{xy} \sin \theta \cos \theta$ ,  $\sigma_T$  equals minus  $2 \tau_{xy} \sin \theta \cos \theta$  and shear stress  $\tau_{LT}$  is equal to  $\tau_{xy} (\cos^2 \theta - \sin^2 \theta)$ .

So, these are the things. So, if this is the case and then we also write now  $\sigma_L$  is going to be a source of  $\epsilon_L$  and  $\epsilon_T$ ,  $\sigma_T$  will also generate  $\epsilon_L$  and  $\epsilon_T$  and  $\tau_{LT}$  will generate a shear strain.

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The image shows a slide with handwritten mathematical derivations. The equations are as follows:

$$\epsilon_L = \frac{\sigma_L}{E_L} - \frac{\nu_{TL}}{E_T} \cdot \sigma_T = 2 \tau_{xy} \sin \theta \cos \theta \left[ \frac{1}{E_L} + \frac{\nu_{TL}}{E_T} \right]$$

$$\epsilon_T = \frac{\sigma_T}{E_T} - \nu_{LT} \cdot \frac{\sigma_L}{E_L} = 2 \tau_{xy} \sin \theta \cos \theta \left[ \frac{\nu_{LT}}{E_L} + \frac{1}{E_T} \right]$$

$$\gamma_{LT} = \frac{\tau_{xy}}{G_{LT}} (\cos^2 \theta - \sin^2 \theta)$$

$$\gamma_{xy} = 2 (\epsilon_L - \epsilon_T) \sin \theta \cos \theta + \gamma_{LT} (\cos^2 \theta - \sin^2 \theta) = \frac{\tau_{xy}}{G_{xy}}$$

$$\frac{1}{G_{xy}} = \frac{1}{E_L} + \frac{1}{E_T} + \frac{2\nu_{LT}}{E_L} - \left[ \frac{1}{E_L} + \frac{1}{E_T} + \frac{2\nu_{LT}}{E_L} - \frac{1}{G_{LT}} \right] \cos^2 2\theta$$

So, we can from these things we can write expressions for  $\epsilon_L$ .  $\epsilon_L$  is what,  $\sigma_L$  over  $E_L$  minus  $\nu_{TL}$  over  $E_T$  times  $\sigma_T$  and from this we get  $2 \tau_{xy} \sin \theta \cos \theta$ , in parentheses  $1$  over  $E_L$  plus  $\gamma_{TL}$  over  $E_T$ .

The negative sign goes away because we have a negative sign here and  $\tau_{xy}$  is also negative. So, that is why it goes away. Similarly,  $\epsilon_T$  is equal to  $\sigma_T$  over  $E_T$  minus  $\nu_{LT}$  times  $\sigma_L$  over  $E_L$  and if we do all the mathematics we get  $2 \tau_{xy} \sin \theta \cos \theta$  minus  $\nu_{LT}$  over  $E_L$  plus  $1$  over  $E_T$  and finally,  $\gamma_{LT}$  equals  $\tau_{xy}$  by  $G_{LT} \cos^2 \theta - \sin^2 \theta$ .

Now, we realize, but if we do the strain transformations calculations we realize that  $\gamma_{xy}$  can be expressed in terms of  $\epsilon_T$ ,  $\epsilon_L$ ,  $\epsilon_{LT}$  using this relation,  $2\epsilon_L \sin\theta \cos\theta - \epsilon_T \sin^2\theta + \gamma_{LT} \cos^2\theta$  minus  $\sin^2\theta$  and it is nothing, but same as  $\tau_{xy} / G_{xy}$ . So, if I combine if we if I put, let us call this as equation E and this is F. So, if I put E in F, what I ultimately get is  $1 / G_{xy} = 1 / E_L + 1 / E_T + 2\nu_{LT} / E_L$  plus  $1 / E_L + 1 / E_T + 2\nu_{LT} / E_L - 1 / G_{xy}$  cosine square  $2\theta$ .

So, this is the last expression and that is where the shear modulus in x y coordinate system. So, this is from the third equation, this is for if we use the equation for  $\gamma_{xy}$ .

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$$\frac{1}{G_{xy}} = \frac{1}{E_L} + \frac{1}{E_T} + \frac{2\nu_{LT}}{E_L} - \left[ \frac{1}{E_L} + \frac{1}{E_T} + \frac{2\nu_{LT}}{E_L} - \frac{1}{G_{xy}} \right] \cos^2 2\theta$$

$$\epsilon_x = -m_x \frac{\tau_{xy}}{E_L} \quad \epsilon_y = -m_y \frac{\tau_{xy}}{E_L}$$

And, if we use the equation for  $\epsilon_x$  and  $\epsilon_y$  we will see from that again we will see that  $\epsilon_x$  is equal to it will get this relation and  $\epsilon_y$  equals minus  $m_y$   $\tau_{xy}$  divided by  $E_L$ , we will get the same relation which we had written earlier. So, mathematically we have shown this.

So, what we have accomplished in last several lectures is we have established concrete mathematical relationships between the four elastic constants which are mutually independent for especially orthotropic plate and we have really expressed all other elastic

constants  $E_x$ ,  $E_y$ ,  $m_x$ ,  $m_y$ ,  $\nu_{xy}$ ,  $\nu_{yx}$  and  $G_{xy}$  in terms of these four elastic constants.

Finally, we will do an example. So, that we become conversant with all this and here we will actually calculate strains for this system.

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EXAMPLE

Diagram showing a square element with axes  $L$  and  $T$ . The element is subjected to stresses:  $-3.5$  MPa (normal stress on  $L$ -faces),  $7$  MPa (normal stress on  $T$ -faces), and  $1.4$  MPa (shear stress on all faces). The  $X$  and  $Y$  axes are also shown, with  $X$  horizontal and  $Y$  vertical.

GIVEN

$E_L = 14$  GPa  
 $E_T = 3.5$  GPa  
 $G_{LT} = 4.2$  GPa  
 $\nu_{LT} = 0.4$      $\nu_{TL} = 0.1$

FIND  $\sigma_x$      $\epsilon_y$      $\gamma_{xy}$

So, we have, this is my  $x$  axis, this is the  $y$  axis and this is my  $L$  axis, this is my  $T$  axis and this lamina is experiencing several stresses. So, it is experiencing negative  $\sigma_x$  and the value is minus  $3.5$  mPa, it is experiencing a positive  $\sigma_y$  and this values  $7$  mPa and finally, it is experiencing shear stress and this value is  $1.4$  mPa.

So, given this stress state and the material properties which are  $E_L$  is equal to  $14$  GPa,  $E_T$  is equal to  $3.5$  GPa,  $G_{LT}$  equals  $4.2$  GPa and  $\nu_{LT}$  is equal to  $0.4$  and  $\nu_{TL}$  is equal to  $0.1$ . So, these are the five material properties we already know in the  $L T$  plane. So, this is given we have to find all the stresses and strains. So, we have to find  $\epsilon_x$ ,  $\epsilon_y$ ,  $\gamma_{xy}$ .

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The slide shows a stress state diagram of a square element with diagonal hatching. The element is subjected to normal stresses of  $-3.5$  MPa on the vertical faces and  $7$  MPa on the horizontal faces. Shear stresses of  $-1.4$  MPa are shown on all four faces. The coordinate system has  $x$  pointing right and  $y$  pointing down.

Material properties and transformation parameters are listed as follows:

- $G_{LT} = 4.2$  GPa
- $\nu_{LT} = 0.4$
- $\nu_{TL} = 0.1$
- Find  $\sigma_x$ ,  $\epsilon_y$ ,  $\gamma_{xy}$

The stress state is summarized in a box:

$$\sigma_x = -3.5 \quad \sigma_y = 7 \quad \tau_{xy} = -1.4$$

Material properties are given as:

$$E_x = 5.02 \text{ GPa} \quad E_y = 10.87 \text{ GPa} \quad G_{xy} = 2.7 \text{ GPa}$$

Transformation parameters are calculated as:

$$\frac{\nu_{xy}}{E_x} = \frac{\nu_{yx}}{E_y} = -0.00446 \quad m_x = 1.8333 \quad m_y = 0.765$$

So, how do we do this? So, first we identify what are the values of sigma x, sigma y and tau xy. So, sigma x is how much minus 3.5, based on the picture sigma y equals 14 and tau xy equals.

Student: Sigma y is equal to 7.

Excuse me, it is 7 here and tau xy is minus 1.4. So, it is important, here the stresses shear stress is negative. So, this is the stress state from the material properties and theta is given to be 60 degrees. So, given this value of theta we first calculate E x, E y, E xy, G xy and so on and so forth. So, we find that E x is equal to 5.02 GPa, E y is equal to 10.87 GPa, G xy equals 2.7 GPa, nu yx, excuse me, I will write that in separate thing, nu xy divided by E x is equal to nu yx by E y and that comes out to be minus 0.00446, m x is equal to 1.8333, m y is equal to 0.765.

So, these are the values we have calculated using the relations earlier and now, from this E x, E y all these properties we can calculate epsilon x.



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The image shows a digital whiteboard with handwritten mathematical derivations. At the top, a box contains the stress components:  $\sigma_x = -2.5$ ,  $\sigma_y = 7$ , and  $\tau_{xy} = -1.4$ . Below this, the material properties are listed:  $E_x = 5.02 \text{ GPa}$ ,  $E_y = 10.87 \text{ GPa}$ , and  $G_{xy} = 2.7 \text{ GPa}$ . The next line shows the calculation of the Poisson's ratio  $\nu_{xy}$  as  $\frac{\nu_{yx}}{E_x} = -0.00446$ , along with  $m_x = 1.833$  and  $m_y = 0.765$ . Finally, the strain components are calculated:  $\epsilon_x = -483$ ,  $\epsilon_y = 705$ , and  $\gamma_{xy} = -443$ . These three values are grouped by a bracket and multiplied by  $10^{-6}$  to indicate they are in micro-strains. The whiteboard interface includes a toolbar at the top and a status bar at the bottom showing '31 / 47'.

So, epsilon x comes to be minus 483, epsilon y comes to 705 and gamma xy comes to minus 443 and all this times 10 to the power of minus 6 micro strains.

So, this is how we do the computation and I hope you will find these lectures very useful in terms of computing the strains if you know the stresses in an x, y plane for a generally orthotropic plate. Next week onwards we will develop this theory further and we will start working on a formulation which will help us analyze not just single layers, but actually laminates of entire systems. So, that concludes our discussion for today and I look forward to seeing you tomorrow.

Thank you.