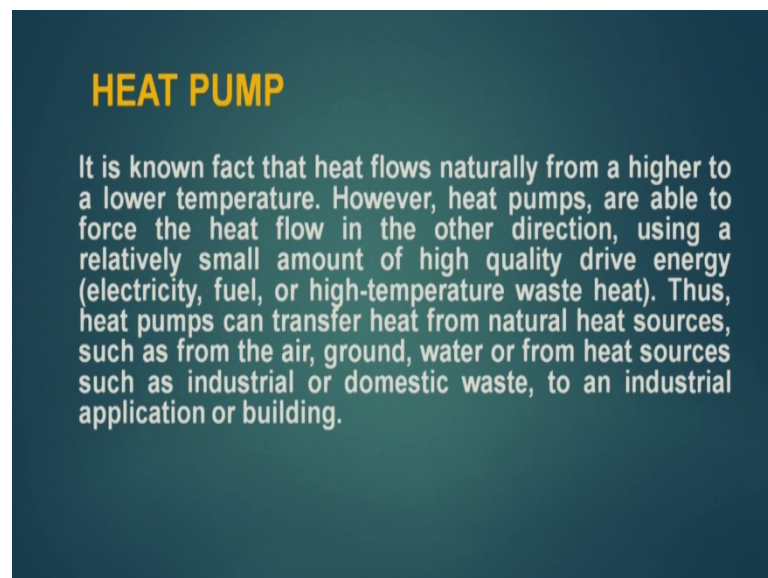


Process Integration
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Module - 06
Integration and Placement of Equipment
Lecture - 03
Integration of heat pump

Welcome to the lecture series on process integration, this is module six lecture number three topic of today lecture is integration of heat pump, what is the heat pump. Let us see it is a known fact that it flows naturally from higher to a lower temperature.

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However, heat pumps are able to force the heat flow in the other direction, that is lower to higher temperature using a relative small amount of high quality drive energy. This can be electricity fuel or high temperature waste heat, thus heat pumps can transfer heat from natural heat sources such as from air ground water or from heat sources, such as industrial or domestic wastes to an industrial application or to a building. So, in sort heat pump works in opposite direction of the natural flow of heat, that is it takes it from a lower temperature and passes it at a higher temperature.

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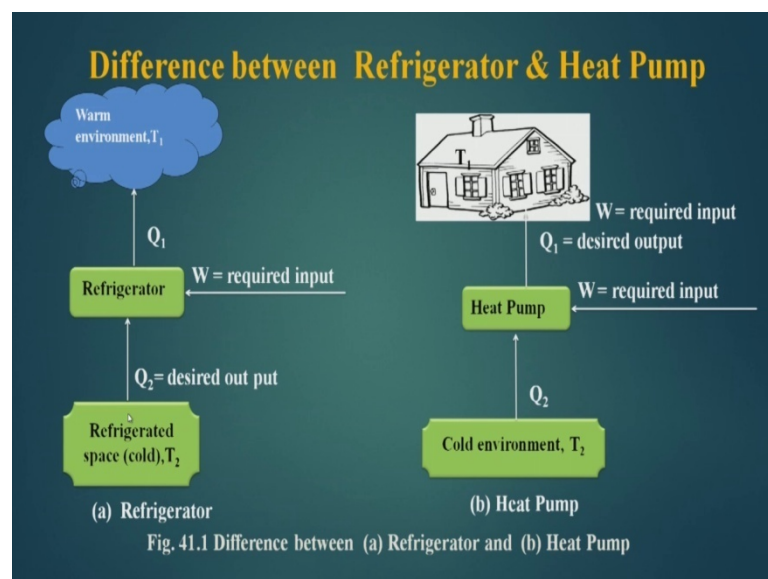
HEAT PUMP

Heat pumps can also be used for cooling purposes also. For such a situation, heat is transferred in the opposite direction i.e. from the application that needs to be cooled to surroundings at a higher temperature where heat is rejected.

In short, the objective of a refrigerator is to remove heat from the cold medium whereas, the objective of a heat pump is to supply heat to a warm medium.

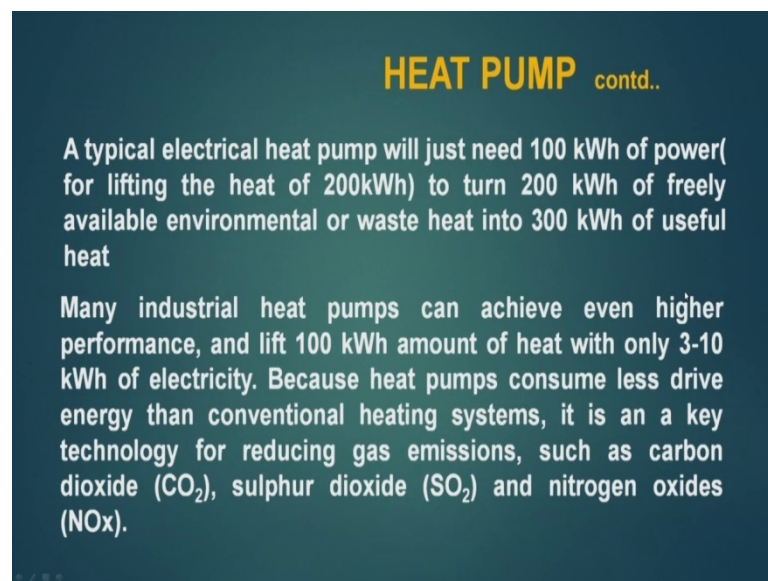
Heat pumps can also be used for cooling purposes. For such a situation, heat is transferred in the opposite direction from the application, that needs to be cooled to surroundings at a higher temperature, where heat is rejected. In short, the objective of a refrigerator is to remove heat from the cold medium whereas, the objective of a heat pump is to supply heat to a warm medium. So, this difference though technically, they are same, but as far as utility is concerned the refrigerator and heat pump have different utilities.

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Now let us see what is the difference between refrigerator and heat pump pictorially. Now here the refrigerator, it takes of heat from a refrigerated space. It is cold at T_2 temperature and fix of Q_2 amount of heat, and then posses to environment, which is at a higher temperature at T_1 , but the amount of heat is Q_1 . So, there is a difference between Q_1 and Q_2 . Q_1 is more than Q_2 , and to transfer this heat from cold to the hot we required W amount of work, which is converted into heat and adds up to this. That means, Q_2 is less than Q_1 , but here the job is the same it takes of it from a cold environment which is at T_2 fix of heat Q_2 . And then discharges to a say house to warm it the amount of heat is Q_1 and to do this it spends W amount of heat, but the desired output is here Q_1 , but here the desired output is Q_2 . So, when it is working in a refrigeration amount the desired output is Q_2 , and when it is working as a heat pump amount its desired output is Q_1 though the technology involved in doing both the jobs are almost same.

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HEAT PUMP contd..

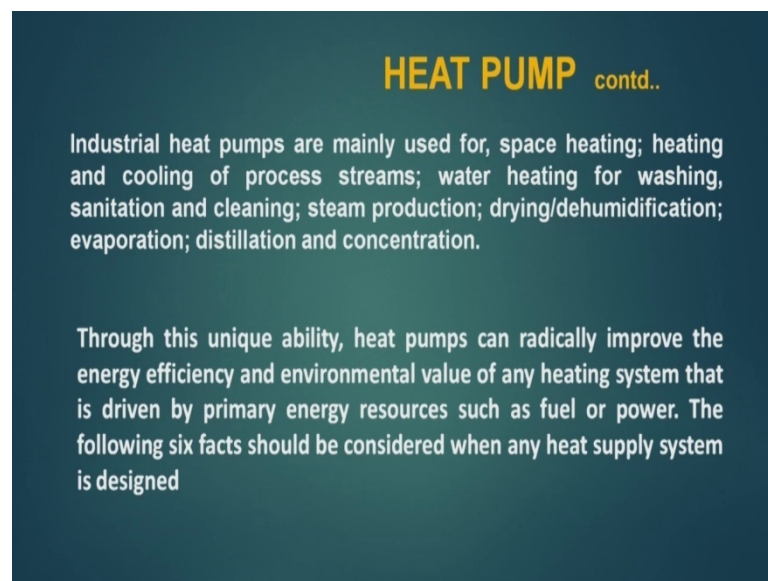
A typical electrical heat pump will just need 100 kWh of power (for lifting the heat of 200kWh) to turn 200 kWh of freely available environmental or waste heat into 300 kWh of useful heat

Many industrial heat pumps can achieve even higher performance, and lift 100 kWh amount of heat with only 3-10 kWh of electricity. Because heat pumps consume less drive energy than conventional heating systems, it is an a key technology for reducing gas emissions, such as carbon dioxide (CO_2), sulphur dioxide (SO_2) and nitrogen oxides (NO_x).

Now let us see why the heat pumping is required, why it is required to pick up the low graded heat and convert into a high graded heat and use it, why this is required. A typical electrical heat pump will just need lift 100 kWh of power for lifting, the heat of 200 kWh to turn 200 kWh of freely available environmental or waste heat into 300 kWh of useful heat. That means, while picking up a heat of 200 kWh, we have to spend around 100 kilo kWh to pump it to a higher temperature level, and that higher temperature level how much to we get, it is about 300 kWh.

Many industrial heat pumps can achieve even higher performance, and lift a 100 kWh amount of heat with only 3-10 kWh of electricity because heat pumps consume less drive energy than conventional heating systems. It is an it is a key technology for reducing gas emissions such as carbon dioxide sulphur dioxide and nitrogen oxides, because more we consume electricity or more we do the heating part of it. So, the carbon dioxide will be generated and sulphur dioxide will be generated nitrogen oxides will be generated. So, heat pump will provide heat energy at a cheaper cost also to some extend and it saves the environment from pollution.

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Industrial heat pumps are mainly used for space heating and cooling of process streams water heating or washing sanitation, and cleaning steam production drying dehumidification evaporation distillation and concentration. So, a large area of technology is serviced by this heat pumps though this is the unique technique, and it has got unique ability heat pumps can radically improve the energy efficiency, and environmental values of any heating systems. That is driven by primary energy resources such as fuel or power, the following six factors should be considered when any heat supply system is designed. So, if I am designing a heat supply system by heating it by a fuel. So, I should consider all these things six points before planning a heat supply system.

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- Direct combustion to produce heat is never the most efficient use of fuel as it creates pollution.
- Heat pumps are more efficient because they use renewable energy in the form of low-temperature heat.
- If the fuel used by conventional boilers were redirected to supply power for electric heat pumps, about 35-50% less fuel would be needed, resulting in 35-50% less emissions.
- Around 50% savings are made when electric heat pumps are driven by CHP (combined heat and power or cogeneration) systems.
- Whether fossil fuels, nuclear energy, or renewable power is used to generate electricity, electric heat pumps make far better use of these resources than do resistance heaters.
- The fuel consumption, and consequently the emissions rate, of an absorption or gas-engine heat pump is about 35-50% less than that of a conventional boiler.

What are those direct combustion to produce heat is never the most efficient use of fuel as it creates pollution. The second point is heat pumps are most efficient, because they use renewable energy in the form of low temperature heat. The third point is if the fuel used by conventional boilers were redirected to supply power for electric heat pumps, about 35-50 percent less fuel would be needed resulting in 35-50 percent less emission. The fourth point is around 50 savings are made, when electric heat pumps are driven by CHP, that is combined heat and power cogeneration system. The fifth point is whether fossil fuels nuclear energy or renewable energy is used to generate electricity electric heat pumps make far better use of these resources. Then do resistance heaters the sixth point is the fuel consumption and consequently the emission rates of an absorption or gas engine heat pump is about 35-50 percent less than that of a conventional boiler.

So, these points should be taken into account, when we are considering a heating system if the temperature lift is low may be that most suitable form of heating will be using heat pumps. Now the heat pump is a reverse form of heat engine a heat engine running in reverse direction is basically, a heat pump from the first law of thermodynamics. The W -work will be Q_1 into n_c the carbon efficiency by n_m mechanical, that is $Q_1 T_1$ minus T_2 divided by n_m mechanical into T_1 the overall efficiencies of heat engine, and heat pumps are W into heater into Q_1 .

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Coefficient of performance for heat pump

$$W = \frac{Q_1}{COP_{HP}} = \frac{Q_2}{COP_R}$$

$$COP_{HP} = \frac{\text{Desired output}}{\text{Required output}}$$

$$= \frac{\text{Heating Effect}}{\text{Work input}} = \frac{Q_1}{W} = \frac{1}{\eta}$$

$$= \frac{\eta_{mech} T_1}{(T_1 - T_2)}$$

$$COP_R = \frac{\text{Desired output}}{\text{Required input}} = \frac{Q_2}{W}$$

$$= \frac{Q_1 - W}{W} = COP_{HP} - 1$$

Hence, the heat pump as a low temperature lift ($T_2 \rightarrow T_1$) gives a high COP_{HP} and a large amount of upgraded heat per unit power.

Closed cycle heat pump

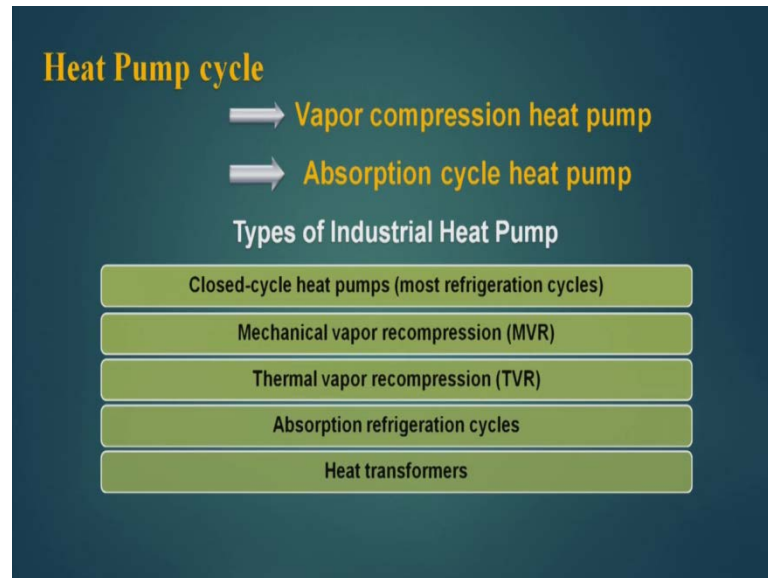
And from the first law of thermodynamics W is equal to Q_1 minus Q_2 , where for heat engines the converse efficiency is greater than the η for heat engines, and it is η is greater than η_c for heat pumps. Now this heat pumps will be coefficient of performance and W is equal to Q_1 divided by COP_{HP} , that is for heat pump and Q_2 is equal to COP_R for refrigerator COP_R stands for refrigeration COP_{HP} . That is for heat pump is a desired output divided by required output heating effect by work input Q_1 , W is equal to 1 by η and is equal to $\eta_{mechanical} T_1$ minus T_2 . And COP_R for refrigeration is desired output which is Q_2 in this case and required input is W So, if I find out the relationship between COP_R and COP_{HP} which is COP_R is equal to $COP_{HP} - 1$.

Hence the heat pumps of the heat pump as low temperature lift you want T_2 tends to T_1 gives high COP ; that means, if lift is less this COP of heat pump is high and large amount of upgraded heat per unit of power. We will get now how the heat pump works let us say evaporator here, that is a liquid represent which takes of heat at a certain lower temperature T_2 it takes of Q_2 amount of heat, and then this evaporates here the vapors are pump the vapors are compressed and some external work W is given here.

The compressed vapors are cool down in a condenser, where it rejects it at a T_1 temperature, which is evaporator the T_2 temperature and the rejection is Q_1 , which is more than the Q_2 and then this liquid goes here across with throttle valve where the pressure is reduced. And this converts into a vapor and liquid mixture and goes into the

evaporator, this is a cycle and this cycle goes on, and this is how the heat pump takes of heat at a lower temperature the Q_2 amount of heat and rejects Q_1 amount of heat is more than Q_2 at a higher temperature.

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Now, that can be many type of heat pump cycles to measure cycles are vapor compression heat pump, and the other is absorption cycle heat pump the different types of the industrial heat pumps are closed cycle heat pumps mostly refrigeration cycles mechanically vapor compression MVR. Thermal vapor compression TVR and then absorption refrigeration cycles and then heat transformers. So, these are the different type of heat pumps, they have different COP values different efficiency values and they are used in different places the fourth is reverse Brayton cycle heat pumps.

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Closed-cycle heat pumps (most refrigeration cycles):

A great majority of heat pumps work according to this principle. A volatile liquid, known as the working fluid or refrigerant, circulates through the components of this cycle. The main components in such a heat pump system are the compressor, the expansion valve and two heat exchangers referred to as evaporator and condenser. The components are connected to form a closed circuit. It takes in heat and evaporates in the evaporator, is compressed and then condensed to give out heat at a higher temperature at condenser, and returned to the evaporator via a expansion valve which cools the refrigerant to evaporator temperature

Now, let us see these different heat pumps. How they and work what are that way of working the closed cycle heat pumps mostly refrigeration cycles a great majority of heat pumps work according to this principle, which we have also explained a volatile liquid known as the working fluid or refrigerant circulates through the components of this cycle. The main components in such a heat pump system are the compression the expansion valve, and two heat exchangers referred to as evaporator and condenser the components are connected to form a closed circuit. It takes heat and evaporates in the evaporator then the vapor is compressed, and then condensed to give out heat at a higher temperature at the condenser and returned to the evaporator via a expansion valve, which cools the refrigerant to evaporator temperature. So, this is the philosophy or this the working principle of a closed cycle heat pump, now in a mechanical vapor rep recompression unit which is called short form is MVR.

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Mechanical vapour recompression (MVR):

Categorized as open or semi-open heat pumps. In a open systems, vapour from an industrial process is compressed by a compressor(driven by electricity or a plant turbine) to a higher pressure (thus a higher temperature) and condensed in the same process giving off heat. In semi-open systems, heat from the recompressed vapour is transferred to the process via a heat exchanger. Because one or two heat exchangers are eliminated (evaporator and/or condenser) and the temperature lift is generally small, the performance of MVR systems is high, with typical coefficients of performance (COPs) of 10 to 30. Current MVR systems work with heat-source temperatures from 70-80°C, and deliver heat between 110 and 150°C, in some cases up to 200°C. Water is the most common 'working fluid' (i.e. recompressed process vapour), although other process vapours are also used, notably in the petro- chemical industry

That is categorized as open or semi open heat pump in a open system vapor from an industrial process is compressed by a compression. This compression is either driven by electricity or plant turbine to a higher pressure. Thus a higher temperature and condensed in the same process giving of heat in semi open system heat from the recompressed vapor is transferred to a process via a heat exchanger, because one or two heat exchangers are eliminated evaporator or condenser or and condenser. And the temperature lift is generally small the performance of MVR system is high with typical COP values ranging from 10-30, which is a very high value current MVR systems work with heat source temperatures from 70-80 degree centigrade and deliver heat between 110 and 150 degree centigrade. In some cases up to 200 degree centigrade water is the most common working fluid although other process vapors are also used notably in the petro chemical industry.

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Thermal vapour recompression (TVR):

High-pressure steam is passed into a venturi-type thermocompressor, and mixed with lower-pressure steam to give a larger flow at an intermediate temperature and pressure. This also includes ejectors, mainly used for drawing a vacuum

Absorption refrigeration cycles:

These refrigeration cycles take waste heat above ambient and extract some below ambient heat from the process and rejects all heat at a medium temperature close to ambient. These are not widely used in industrial applications. Current systems with water/lithium bromide as working pair achieve an output temperature of 100°C and a temperature lift of 65°C. The COP typically ranges from 1.2 to 1.4. The new generation of advanced absorption heat pump systems will have higher output temperatures (up to 260°C) and higher temperature lifts

Now, thermal vapour compression which is called TVR high pressure steam is passed into a venturi type thermo compression, and mixed with the low pressure steam to give a larger flow at an intermediate temperature and pressure. This also includes ejectors mainly used for drawing a vacuum the absorption refrigeration cycle. These refrigeration cycles take waste heat above ambient and extract some below ambient heat from the process, and rejects all heat at a medium temperature close to ambient. These are not widely used in industrial applications current systems with water and lithium bromide as working pair achieve an output temperature of about 100 degree centigrade. And a temperature lift of about 65 centigrade, the COP typically ranges from 1.2 to 1.4 this is very low sloppy as we have seen from mechanically vapour recompression systems. The new generation of advanced absorption heat pump systems will have higher output temperatures up to 250 degree centigrade and higher temperature lifts.

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Heat transformers :

These have the same main components and working principle as absorption heat pumps. With a heat transformer waste heat can be upgraded, virtually without the use of external drive energy. Waste heat of a medium temperature (i.e. between the demand level and the environmental level) is supplied to the evaporator and generator. Useful heat of a higher temperature is given off in the absorber. All current systems use water and lithium bromide as working pair. These heat transformers can achieve a delivery temperatures up to 150°C, typically with a lift of 50°C. COPs under these conditions range from 0.45 to 0.48. In short, heat transformers take in waste heat, upgrade some of it to a useful temperature and cool the rest, thus acting as "heat splitters". They are in effect a reversed absorption refrigeration cycle working entirely above-ambient temperature

Now, that is another category of heat pump which is called heat transformer. Though its COP is very less these have the same main components, and working principle as absorption heat pumps with a heat transformer. Waste heat can be upgraded virtually without the use of external drive energy waste heat of a medium temperature is supplied to the evaporator. And generator useful heat to a higher temperature is given up in the absorber, all current systems use water and lithium bromide as working pair. These heat transformers can achieve a delivery temperature up to 150 degree centigrade typically with a lift of 50 degree centigrade also COP under these conditions range from 0.45 to 0.48, which is a very low value in short heat transformers take in waste heat upgrade some of it to a useful temperature and cool the rest. That means, whatever heat it picks up some part of heat is upgraded and some part of heat is degraded, thus acting as a heat splitter they are in effect a reversed absorption refrigeration cycle working entirely above ambient temperature.

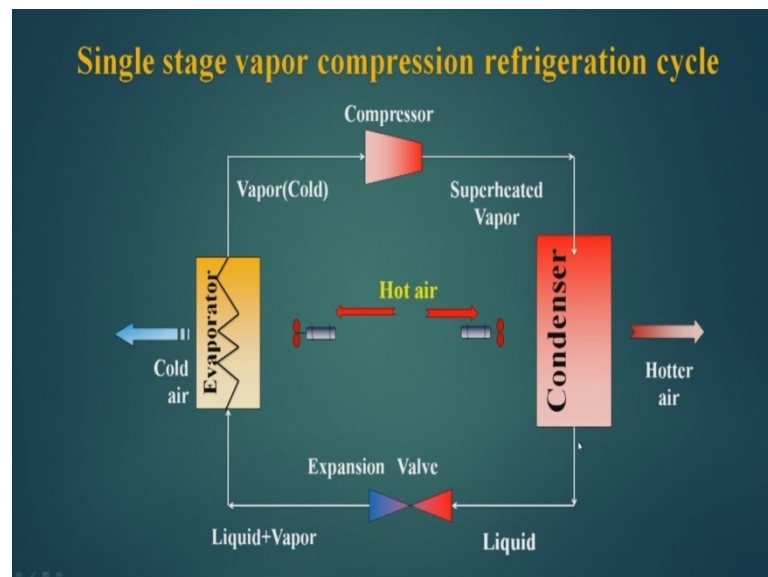
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Reverse Brayton-cycle heat pumps:

It recovers solvents from gases in many industrial processes. Solvent loaded air is compressed, and then expanded. The air cools through the expansion, and the solvents condense and are recovered. Further, expansion takes place in a turbine, which drives the compressor.

Reverse Brayton-cycle heat pumps it recovers solvent this pump recovers solvents from gases in many industrial processes solvent loaded air is compressed, and then expanded the air cools through the expansion, and the solvent condense and are recovered further expansion takes place in a turbine which drives the compression.

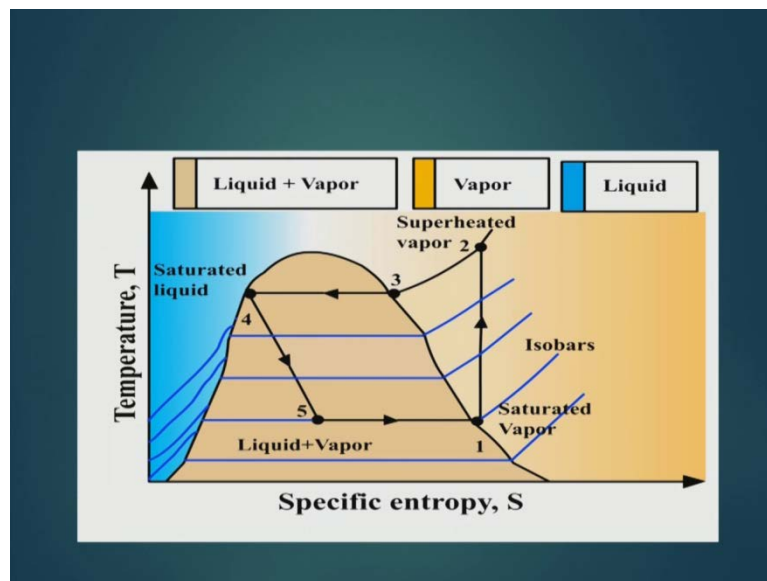
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So, after seeing the different type of heat pumps let see the compression refrigeration and the absorption refrigeration which is a compression refrigeration cycle, this is the evaporator in which the liquid and vapour mix is evaporated. So, it cools and the liquid

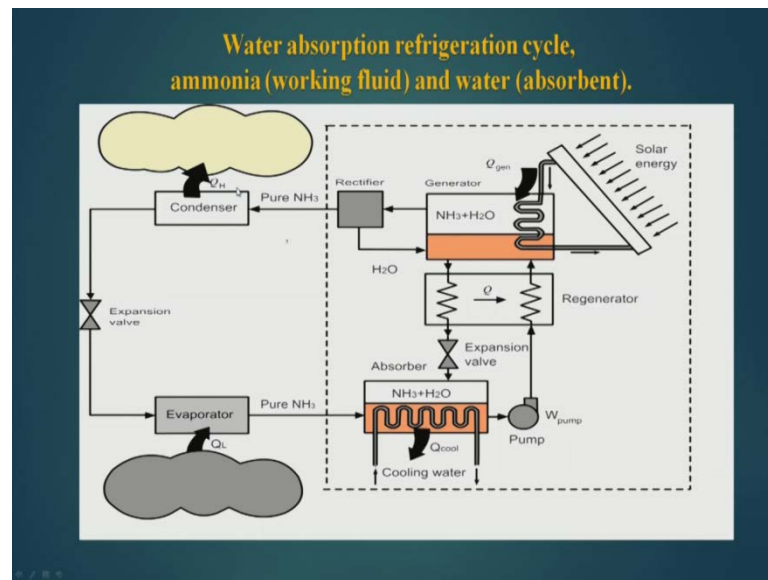
vapor goes to here, it cools down and then vapour is created here. Vapour is compressed, the superheated vapour goes to the condenser. While it condensed, then it goes to the expansion valve, here the expansion takes place, that a special drop express and we get a low pressure liquid and vapour mixture that goes to the evaporator, and the liquid part evaporates here and then we get vapour. So, this is how the compression refrigeration cycle works.

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This is the entropy-temperature, entropy diagram. This is the compression part then superheating then condensing then expansion then vaporization takes place.

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Now this gives you an absorption refrigeration cycle. This is the evaporator where we are using the NH₃ and H₂O combination here. So, here the liquid n h three is evaporated you get a vapour is absorbed in a H₂O, and the absorbed vapour and liquid is pumped through a regenerated to here. The heating takes place the vapours come out with water, they go for rectification. From rectification almost pure NH₃ at a higher pressure goes to the condenser, where it condenses to the liquid and the heat is pumped to the atmosphere, and then expansion takes place. The temperature comes down the liquid, and vapour mixture of N H 3 goes to evaporator where it evaporates basically, and then you will get again pure almost pure NH₃ that is ammonia and then it is being absorbed by H₂O. So, this is shows a absorption refrigeration system and the driving force is the heater here, the solar heater it provides the energy for cooling. So, with low temperature heat these system can work, but you need a large amount of low temperature heat.

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Performance of Heat pump

Heat delivered by the pump = Heat extracted from The heat source + Energy needed to Drive the cycle

Steady-state performance of an electric compression heat pump is given by COP

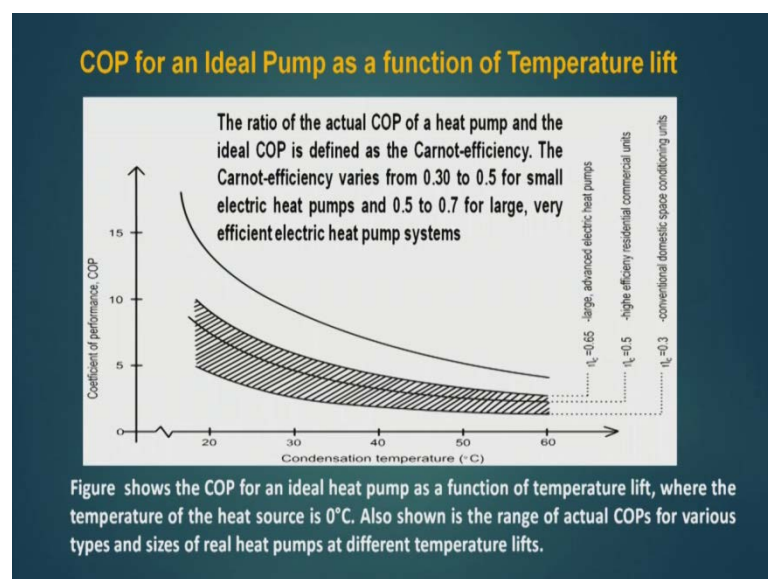
$$COP = \frac{\text{heat delivered by the pump}}{\text{electricity supplied to the compressor}}$$

For engine and thermally driven heat pump performance is indicated by primary energy ratio (PER).

For electrically driven heat pumps a PER can also be defined, by multiplying the COP with the power generation efficiency.

Now, the performance of heat pump the heat delivery by the heat pump is equal to heat extracted from the heat source plus energy needed to drive the cycle. So, this Q_1 is equal to Q_2 plus W . The steady state performance of an electric compression heat pump is given by this COP, which is defined like this heat delivered by the pump divided by the electricity supplied to the compression for engines, and thermally driven heat pump performance is indicated by primary energy ratio PER for electricity driven heat pumps. A PER can also be defined by multiplying this COP with the power generation efficiency.

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Now, this source that with different type of heat pumps or different size of heat pumps how this COP with a changes with the temperature lift, now here this figure shows the COP of an ideal heat pump as a function of temperature lift, where the temperature of the heat source is 0. So, it is picking up heat from a 0 degree heat source and condensing at different temperatures. So, whatever temperature here it is written is practically heat lift because my bottom line is at 0 degree centigrade. So, here we see that the η_c is point 0.65 for large advanced electric heat pumps, and for it is 0.5 for highly efficient residential commercial units and 0.3 for conventional domestic safe conventional units. So, the receiver of the actual COP of a heat pump and the ideal COP is defined as the Carnot efficiency varies from 0.30 to 0.5 for small electric heat pumps and 0.5 to 0.7 for large and very efficient electric heat pump systems. So, here we see that it is the function of the lift as well as size of the heat pump, and the type of the technology which is used for heat pumps.

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COP/PER range for heat pumps with different drive engines

| Heat Pump types(s) | COP | PER |
|------------------------|---------|---------|
| Electric (compression) | 2.5-5.0 | |
| Engine(compression) | | 0.8-2.0 |
| Thermal(Absorption) | | 1.0-1.8 |

Now this shows the COP and PER range for heat pumps with different drive engines. If I take a electric compression is used for heat pump, this COP is 2.5 to 5.0. If I use a engine compression, it is 0.8 to 2.0 p e r and thermal, if I use its very low is 1.0 to 1.8 for others.

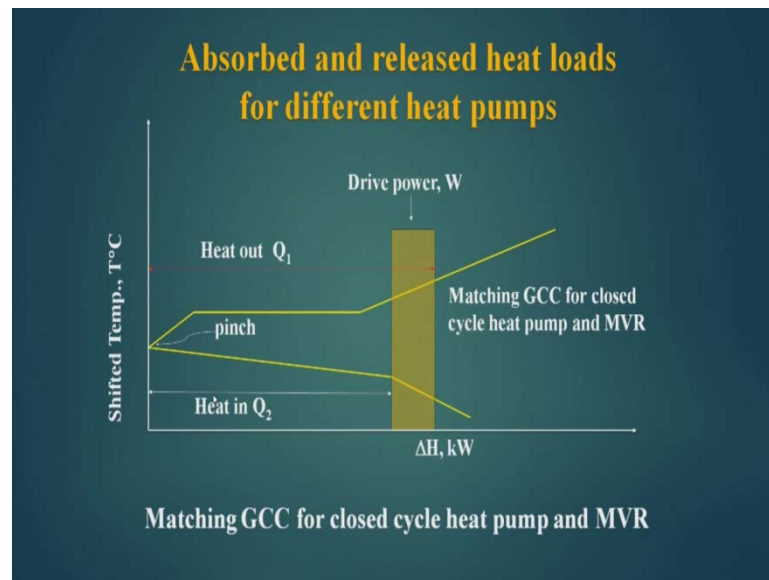
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Technical Characteristics of heat pumps

| System | Delivery Temp., °C | Heat acceptance temp., °C | Temperature Lift, °C | Typical COP |
|-----------------------------|--------------------|---------------------------|----------------------|-------------|
| Electric Compression | | | | |
| R22 | 20-80 | -20-40 | < 60 | 3-5 |
| R12, R500 | 30-95 | -20-65 | <60 | 3-5 |
| R114 | 40-130 | 10-96 | < 60 | 2-4 |
| Mechanical Vapor comp. | >100 | >80 | < 50 | 5-20 |
| Thermal vapor comp. | 60-150 | 45-120 | < 40 | 1.1 |
| Absorption | 30-92 | 5-42 | < 45 | 1.3 |
| Heat Transformer | 80-150 | 58-110 | < 50 | 0.45-0.5 |

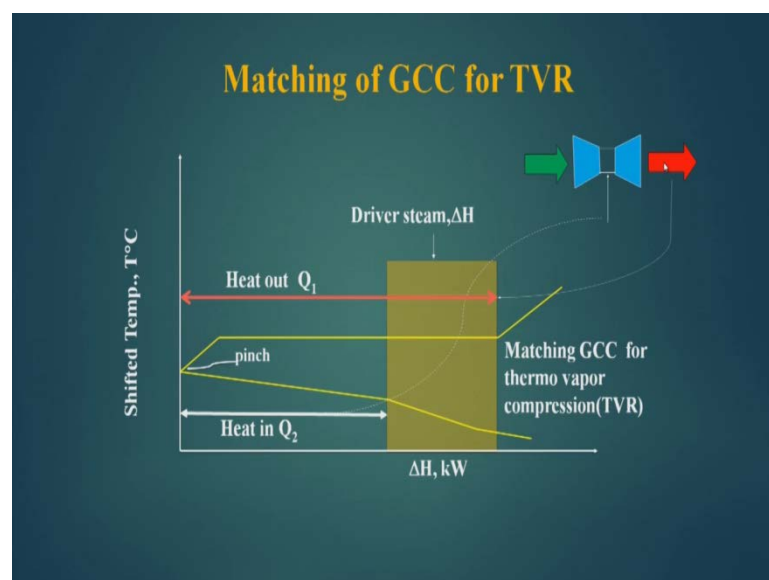
If you see this electrical compression, I am using R 2 2. If I use the delivery temperature is 20 to 80, and the heat acceptance temperature which is basically the evaporated temperature is minus 20 to minus 40, and temperature is lift is about less than 60 degree centigrade, and this COP is 3.5. Similarly for R 1 2 this is 3.5 also. And temperature lift is also same R 1 1 4, this is 60. And this is 224 and mechanical vapour compression, if you take the lift is 50 little bit less, but the COP value is very large 5 to 20. And thermal vapour compression it is 1.1, and lift is also less 40 absorption, it is less than 45 degree and the COP is 1.3. And heat transformer this is less than 50 degree, and if it is very low 0.4 to 0.5. So, here see that mechanical vapour compression the lift is little bit less for this COP is very high.

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Now, if you see that absorbed and released heat loads of different heat pumps, if I am using a closed cycle heat pump. So, this is the amount of Q_2 it picks up the distance between this and this is the amount of Q_2 it picks up, and deliver is from here to here; that means, this band is for driving power. So, this much amount it will pick up and this much amount it will deliver.

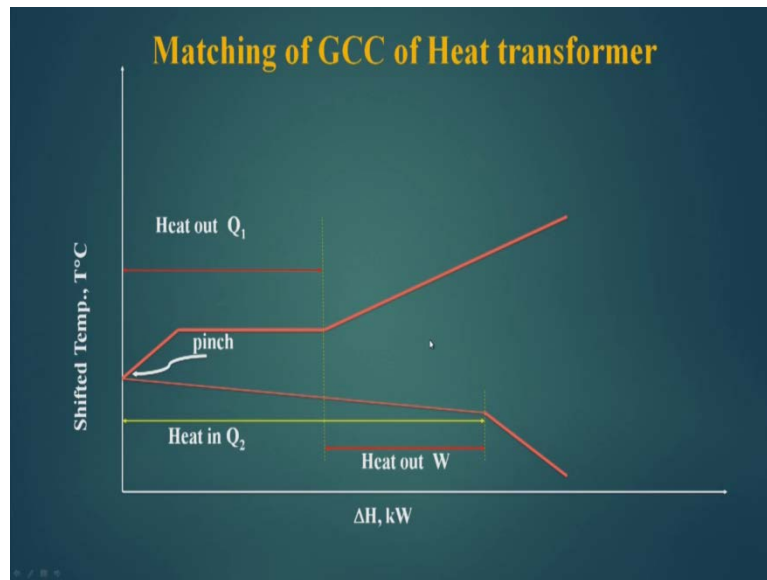
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Now, for thermal vapour compression units this much amount it picks up, and this much amount it delivers and it is the driver steam systems ΔH . So, it adds up to this Q_2

and forms its Q_1 . So, this is amount of heat it delivers and this is amount of heat it picks up. So, this is a TVR, where the high pressure and high temperature steam is injected its struck vapour from here they mix together, and then we get a mix steam which deliver seat.

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This is for heat transformer this is the amount of heat, which is available a part of heat this much is upgraded and this is a downgraded. So, we get less heat here in a heat transformer which is a upgraded heat.

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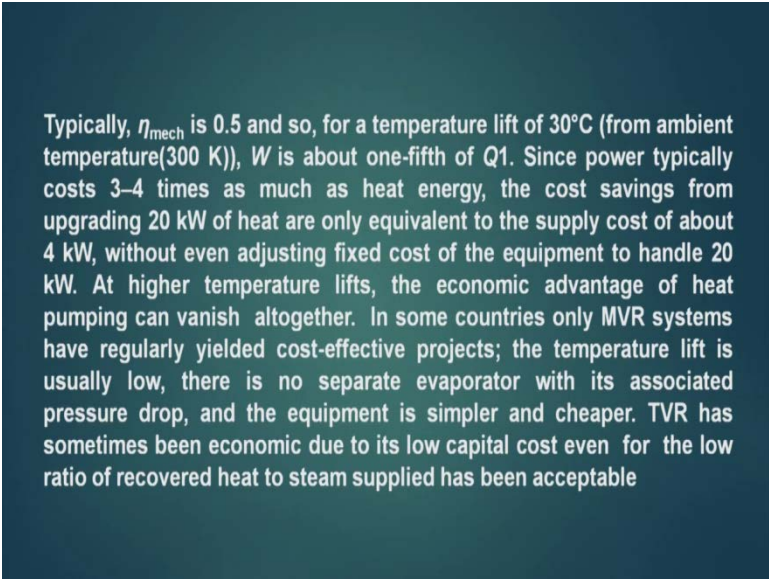
Economics of Heat Pump

- Depends on the temperature lift for which it has to be designed.
- Depends on cost of heat and power which changes very often.

$$W - \frac{Q_1}{COP_P} = \frac{Q_1 (T_1 - T_2)}{\eta_{mech} T_1}$$

Now the economics of heat pump will depend on the temperature lift for which it has to be designed. Because if the temperature lifts are very high, the economy does not support it the economics, and depends on cost of heat and power which changes very often it is a interesting part the cost of heat, and the cost of power is varying. So, is may be possible if the today the heat pump is not economical after few years it will be economical. So, the warden is given by this Q_1 by COP is equal to this is W equal to Q_1 by COP.

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Typically, η_{mech} is 0.5 and so, for a temperature lift of 30°C (from ambient temperature (300 K)), W is about one-fifth of Q_1 . Since power typically costs 3–4 times as much as heat energy, the cost savings from upgrading 20 kW of heat are only equivalent to the supply cost of about 4 kW, without even adjusting fixed cost of the equipment to handle 20 kW. At higher temperature lifts, the economic advantage of heat pumping can vanish altogether. In some countries only MVR systems have regularly yielded cost-effective projects; the temperature lift is usually low, there is no separate evaporator with its associated pressure drop, and the equipment is simpler and cheaper. TVR has sometimes been economic due to its low capital cost even for the low ratio of recovered heat to steam supplied has been acceptable

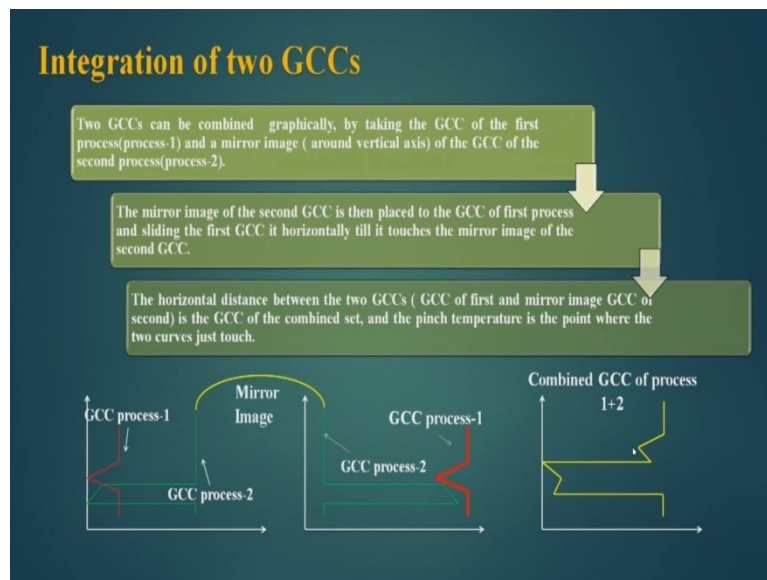
Now to get a fairly good idea how the efficiency of this heat pump works typically the η_{mech} is about 0.5 and. So, for a temperature lift of 30 degree centigrade from ambient temperature W is about 15 of Q_1 ; that means, whatever heat I am delivering its 15 is the power requirement to lift it for about 30 degree centigrade. Since power typically costs 3-4 times as much as the heat energy the cost saving from upgrading 20 KW of heat are only equivalent to this supply cost of about 4 KW without even adjusting fixed cost of the equipment to handle 20 KW at higher temperature lifts. The economic advantage of heat pumping can vanish altogether this we have seen because the economics is a is highly sensitive to the temperature lift.

In some countries, only MVR systems have regularly yielded cost effective projects, because we have seen that the MVR and COP is very high. The temperature lift is usually low, there is no separate evaporator with its associated pressure drop, and the equipment is simpler and cheaper. TVR has sometimes been economic due to its low

capital cost even for the low ratio of recovered heat to steam supplied has been acceptable. So, what we see here the MVR's are very cost effective, but they can only be used or we can say that they are only be economical for lower temperature lift may be below 50 degree centigrate.

Now, let us come to the design part of it for the design we have to integrate this heat pumps with. The GCC. So, the heat pumps has to be described in a shifted temperature and enthalpy diagram as GCC also expressed in a shifted temperature and delta t diagram now let us see how two. GCC's physically when I say two GCC I mean that one GCC for the heat pump, and other GCC for the process now let us see that how to integrate two GCC's.

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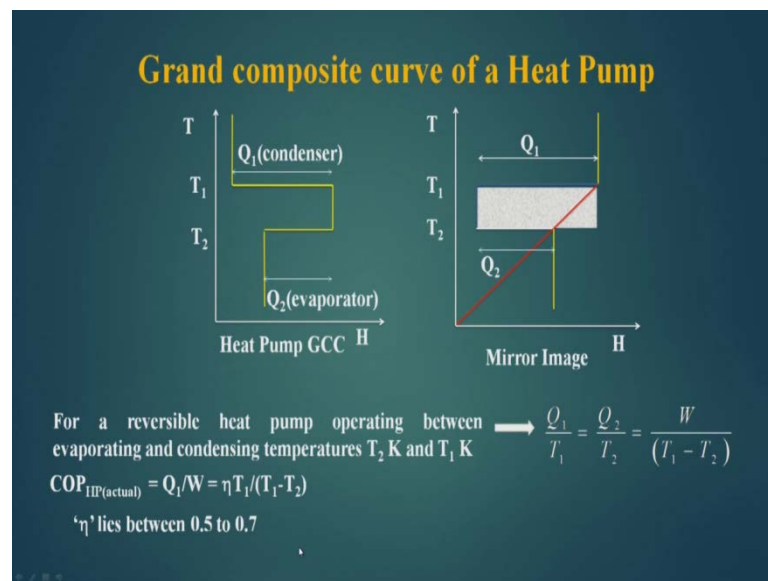


Now, these are two GCC, one GCC for the process and the other GCC for the process two. And basically this will be the heat pump. Now what we have to do we have to find out the mirror image of the second GCC, and then placed to the GCC of the first process and sliding the first GCC to it horizontally till it touches the mirror image of the second GCC. So, if I see the mirror image of the second GCC process here this is this. So, basically on this axis have to rotate it. So, when I rotate it I get this GCC, which is the mirror image of this GCC process two and then I put my first GCC here and this touches at this point. Now the horizontal distance between the two GCC's that is GCC of the first, and mirror image GCC of the second is the GCC of the combined set and the pinch

temperature is the point where the two curves just touch. So, this is the point where it is touching.

So, this will be pinch and the distance between this GCC and this GCC is measured and plotted. So, if I do so my combined GCC of the process one and two will look like this. So, this point here the touching point is this which is the pinch point. So, this pinch point of the combined GCC will be different than the pinch point of the individual GCC, that is process one and process two. Let us see what we have done this two GCC's are to be combined GCC of process one, and GCC of process two then process two GCC has to be its mirror image has to be taken. So, I took the mirror image of the process two GCC here and then slide the process one GCC tively touches at one point and once it touches, I stop it then this gap between this r plotted here that they take different temperature levels. So, I get a combined GCC. Now what I will do with of this combined GCC now, because my aim is to integrate the heat pump with the GCC. So, one of the process of this heat pump say process two and process one is the actual process which I am using.

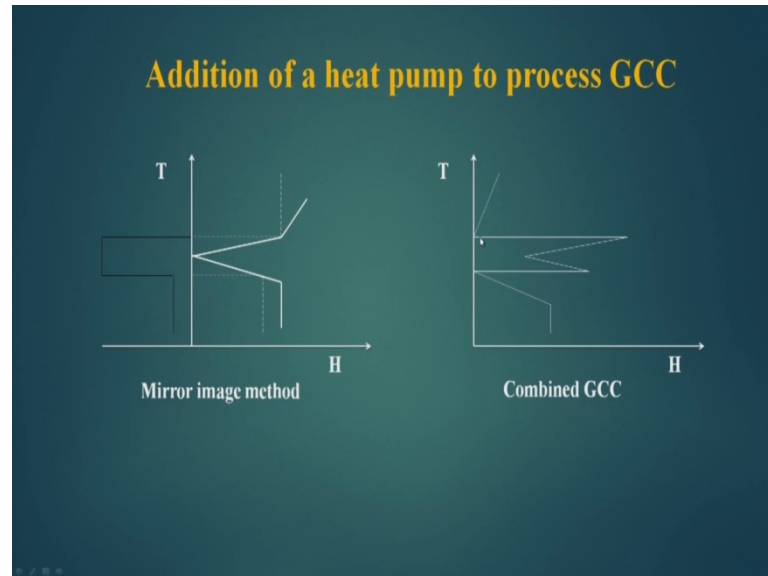
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Now let us see the a heat pump, now the if I plot the heat pump this will look like this this is the evaporator and this is the condenser because the less heat, that is Q_2 will be taken from the evaporator and more evaporator will producing the condenser. So, this look like this. So, if I get a mirror image of this. So, this will look like this now for reversible heat pumps operating between evaporating and condensing temperatures T_2

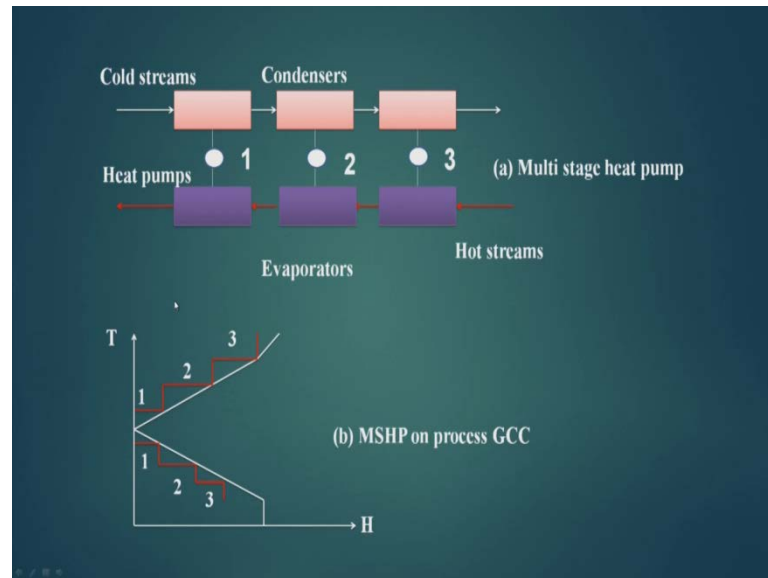
and T_1 . This holds good Q_1 by T_1 is equal to Q_2 by T_2 is equal to W by T_1 minus T_2 and COP of the heat pump actual is this where n lies between 0.5 to 0.7.

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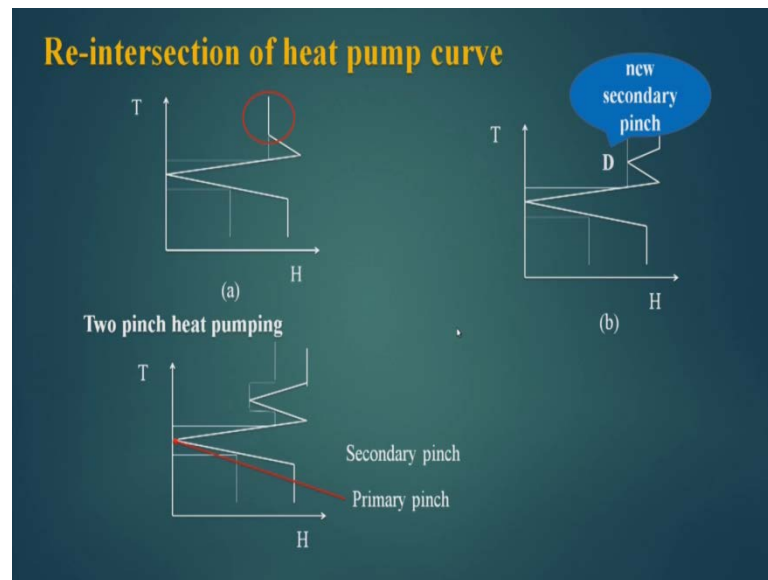
Now let us see how this we have to integrate the heat pump with GCC. So, this is the mirror image of the GCC and this is our process. So, this touches at this two point. So, what I am expecting a two pinch combined GCC, now if I analyze this GCC what I am finding that the heat, which is available here is going down up to this without working no full full work is done. So, at this part I can use a cannot engine to recover some of the energy; that means, the heat pump, which I am using to pump the heat from a lower temperature to a higher temperature is not working very efficiently.

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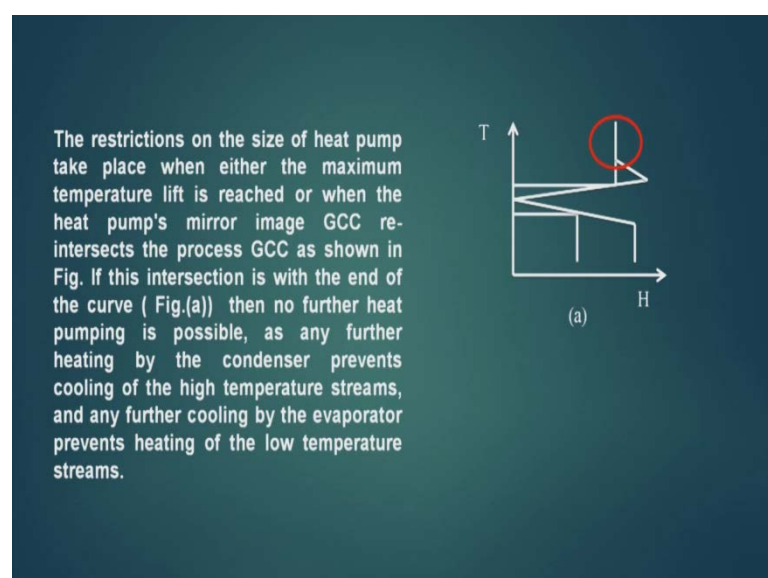
Now, this can be solved to some extent if I am using multi stage heat pumps. So, in the multi stage heat pump will look something like this. And then this is the process GCC, it has got a better fit with multi stage heat pump and; that means, if I am using a multi stage heat pump though the complexity of the design will increase the cost will increase, but it will be more energy efficient than the single stage heat pump. Now this shows that the heat pump is catching this extreme boundary of this GCC of the process. So, this shows the maximum saving which can be done, because if I pump more energy to this then it will start cooling this part of the GCC, which is not required. Because I am using a heat pump to pump energy that means, to cool that to heat the cold process streams and if I increase here the amount; that means, this line will cut it here or in this direction. So, this will start heating this output of the heat pump. So, it is not required.

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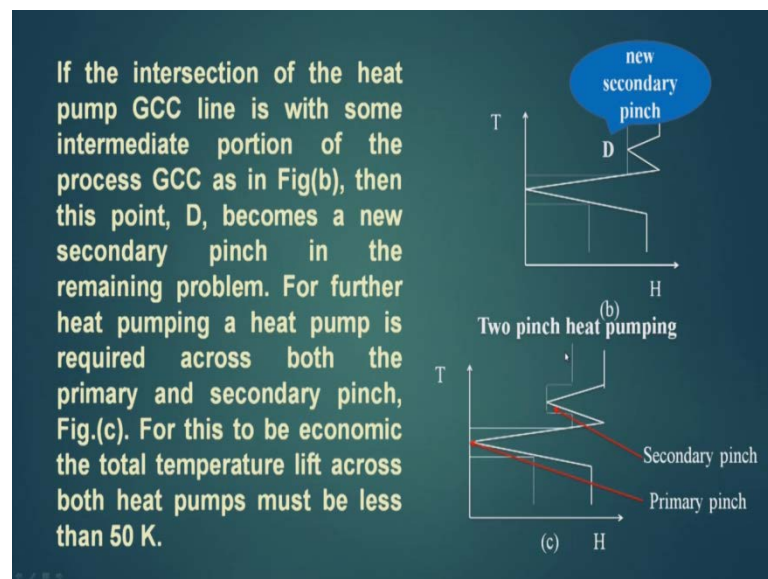
So, this shows the maximum heating which can be done by using a heating pump similarly I cannot increase the condenses also because it will give the reverse effect, but if I have got two pinches here one pinch here and one pinch here this heat can be upgraded. A and can be put here for heating purposes similarly heat available from here can be put here for heating purposes, but in this case there is no scope for improvement, but here there is a case for improvement where the heating can be done. So, here in the third part we can see that we can have this part of heat can pick up and put it here. So, this can be improved.

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Now let us see it. The restrictions on the size of heat pump take place, when either the maximum temperature lift is reached or when the heat pump mirror image GCC re-intersects the process GCC as shown in the figure here. If this intersection is with the end of the curve as shown in the a-part, this is end of the curve this intersection. No further heat pumping is possible as any further heating by the condenser prevents cooling of the high temperature streams, and any further cooling by the evaporator prevents heating of the low temperature streams.

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So, if it touches like this then this is the maximum heating, which we can do if the intersection of the pump GCC is with some intermediate portion of the process GCC as shown in the figure b. Then this point, which is called d point, becomes a new secondary pinch in the remaining problem for further heat pumping. A heat pump is required across both the primary and secondary pinch, because we have seen that the integration of heat pump is beneficial when is integrated across the pinch. So, here across the pinch I am putting of a heat pump. And if is secondary pinch here also I can heat pump across of the pinch, and it is shows in the c part or this will be done. So, here we can employee a secondary pumping as shown here.

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Maximum Economic Lift for a given Heat pump (Saidas M. Ranade, (1988))

$$\frac{T_A}{T_D} \uparrow_D = \left\{ [C_{HPE} + C_{HPC} + C_t + C_c - (\sum \alpha_{H1} C_{H1}) - (\sum \alpha_{CO1} C_{CO1})] \left(\frac{N_t N_M}{H Q_D P} \right) + (MC)_D - N_M (MV)_i - N_t N_M [(MV)_H - (MV)_i] \right\} / [(MC)_D - N_M (MV)_i]$$

Where:

T_A - temperature at which heat is accepted (K)

T_D - temperature at which heat is delivered (K)

C_{HPE} - cost of heat pump evaporator (\$ energy⁻¹)

C_{HPC} - cost of heat pump condenser (\$ energy⁻¹)

C_c - compressor cost (\$ energy⁻¹)

C_t - capital cost of the remaining process (\$ energy⁻¹)

C_{H1} - heater cost (\$ energy⁻¹)

C_{CO1} - cooler cost (\$ energy⁻¹)

α_{H1} - fraction of heater area that can be reused

α_{CO1} - fraction of cooler area that can be reused

N_t - thermodynamic efficiency

N_M - mechanical efficiency

H - hours of operation per year (h yr⁻¹)

P - payback period (yr⁻¹)

Q_D - heat delivered by the heat pump (thermal energy time⁻¹)

$(MV)_H$ - marginal value of the level of thermal energy saved by heat pumping (\$ energy⁻¹)

$(MV)_i$ - marginal value of the waste thermal energy used by the heat pump (\$ energy⁻¹)

$(MC)_D$ - marginal cost of the electricity or shaft power (\$ energy⁻¹)

Saidas M Ranade has derived an equation to give what is the maximum economic lift for a given pump. So, this is the equation which is available where these are the input parameters T_A is the temperature at which heat is accepted, and T_D is the temperature at which heat is delivered. So, if I am able to know this two temperatures I can find out the lift. So, it gives you the ratio when pertinent parameters are entered into this F_b this economic parameter are also there the operating parameters are also there. So, find out a maximum lift for a given heat pump from this and can apply to a problem. So, these are the references.

Thank you.